

## Chapter 3

### 3.1 Stability

- A very important property of a dynamic system is its stability:
  - Internal stability
  - External stability
- *Internal stability* is concerned with the responses of all internal variables (“states”).
- *External stability* studies the input-output behaviour of a dynamic system.

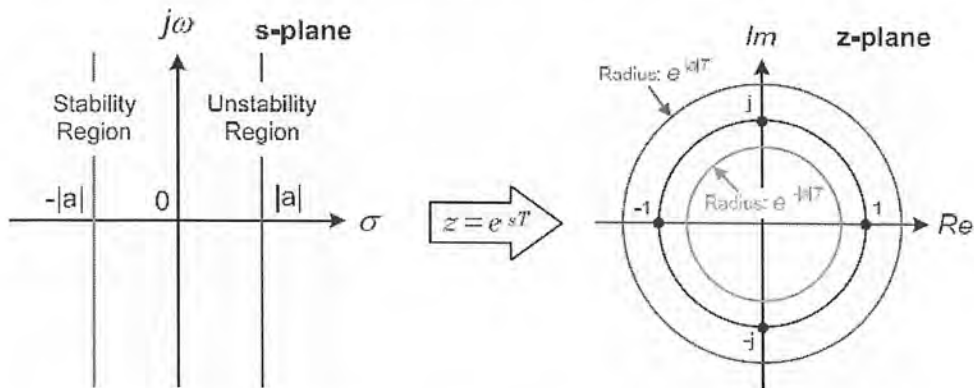
### External Stability

- Most common definition of appropriate response for external stability is that for every *bounded input* (BI), the dynamic system should have a *bounded output* (BO).
- If this condition is satisfied, the system is said to be BIBO stable.

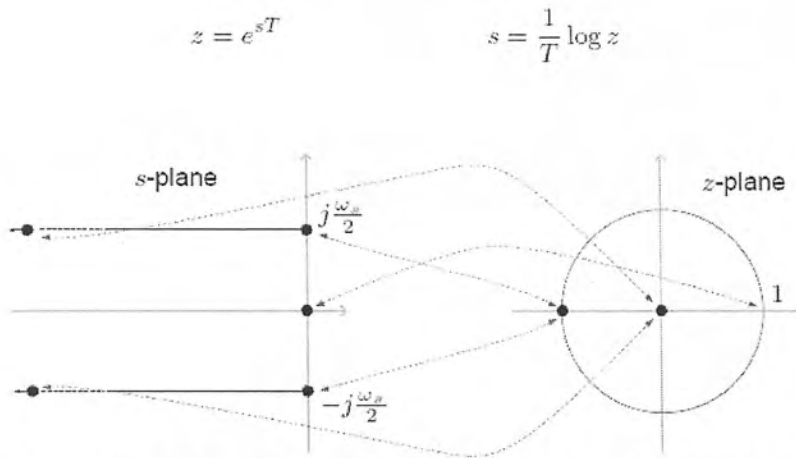
## Stability for Continuous-time Systems

- A system is *asymptotically* stable if it returns to its initial equilibrium state after the application of an *impulse*.
- Stability is a property of the system and it is NOT dependent on a specific input or initial conditions.
- A *continuous-time system* (CTS) is stable if and only if the real parts of its poles are all strictly negative.
  - Poles must be located at left-hand side of the s-plane!

### 3.2 Mapping between s- and z-plane



Let  $s = \sigma + j\omega$  where  $j \hat{=} \sqrt{-1}$   
 Since  $z \hat{=} e^{sT}$   $z = e^{\sigma T + j\omega T} = e^{\sigma T} \cdot e^{j\omega T}$   
 $z = \underbrace{e^{\sigma T}}_{\text{Radius}} (\underbrace{\cos\omega T + j\sin\omega T}_{\text{Depicts unit circle}})$



Correspondence between the primary strip in the  $s$ -plane and the unity circle in the  $z$ -plane.

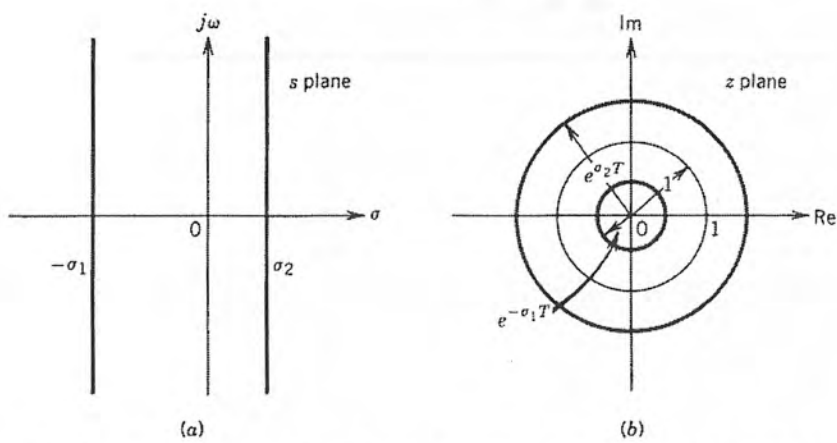
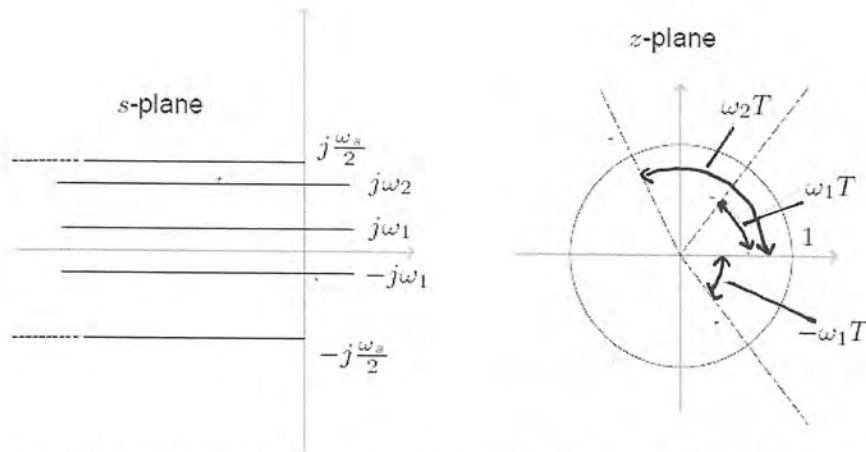


Figure 10.1 (a) Constant attenuation lines in the  $s$ -plane; (b) the corresponding loci in the  $z$ -plane.

$$z = e^{sT}$$

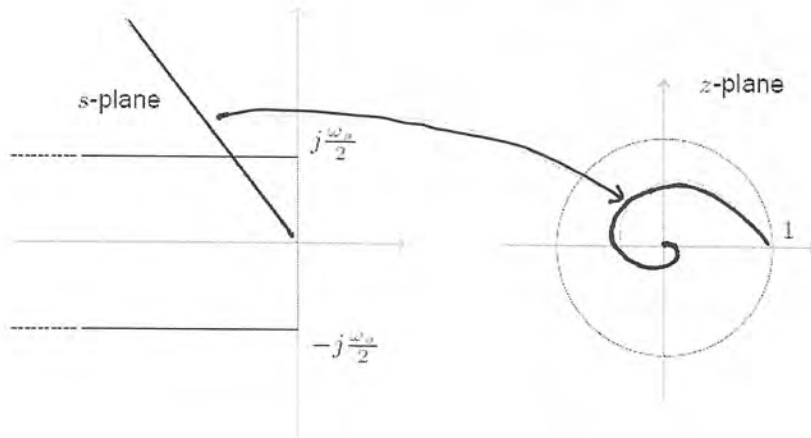
$$s = \frac{1}{T} \log z$$



Constant frequency lines:  $s = \sigma + j\omega$ , with  $\omega$  constant,  $|z| = e^{\sigma T} e^{j\omega T}$ .

$$z = e^{sT}$$

$$s = \frac{1}{T} \log z$$



Constant damping-ratio lines:  $s = -\zeta\omega_n + j\omega_n\sqrt{1-\zeta^2}$ , with  $\omega_n > 0$ ,  $0 < \zeta < 1$  constant,  
 $|z| = e^{-\zeta T\omega_n}$ ,  $\angle z = \sqrt{1-\zeta^2}\omega_n T$ . The locus on the z-plane is a logarithmic spiral.

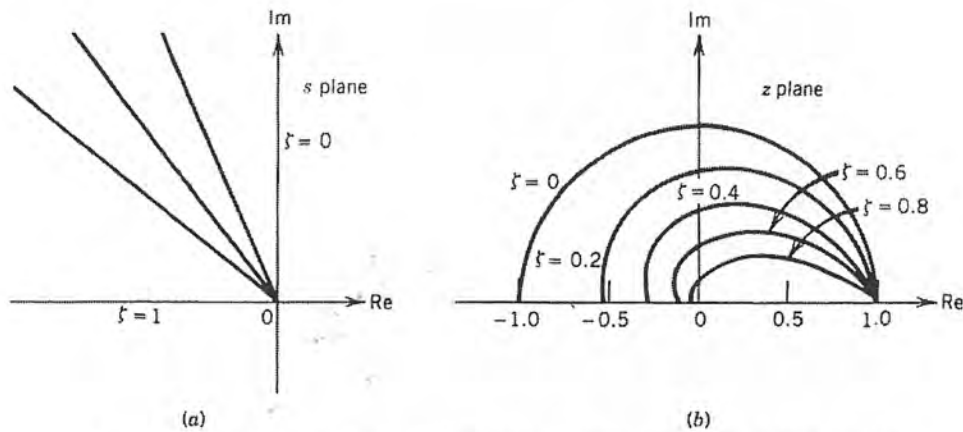
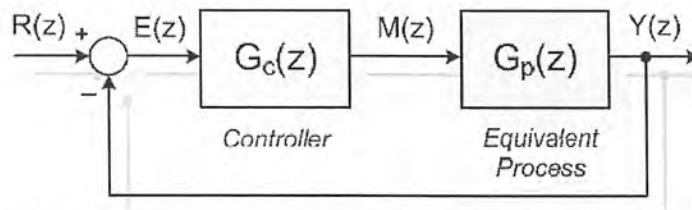


Figure 2.2 (a) Constant  $\zeta$  loci in the  $s$ -plane; (b) constant  $\zeta$  loci in the  $z$ -plane.

## Stability of Control Systems

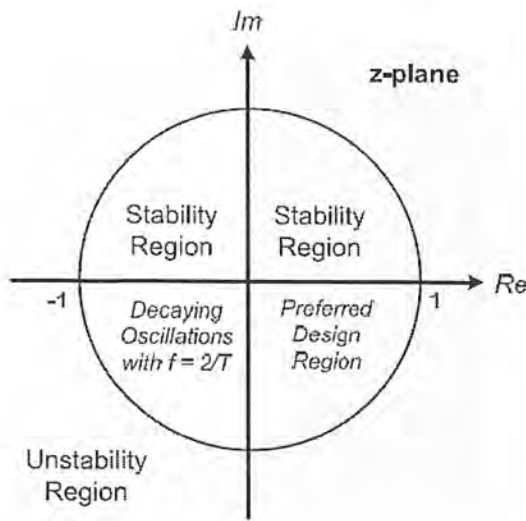


Consider the transfer function of the closed-loop control system shown:

$$\frac{Y(z)}{R(z)} = \frac{G_c(z)G_p(z)}{1 + G_c(z)G_p(z)} = \frac{B(z)}{A(z)} \quad \text{where} \quad \begin{aligned} A(z) &= \text{num}\{1 + G_c(z)G_p(z)\} \\ A(z) &= a_N z^N + \dots + a_1 z + a_0 \end{aligned}$$

$$\text{Hence, } A(z) = \sum_{n=0}^N a_n z^n = \underbrace{\left[ \prod_{i=1}^{N_r} (z + r_i) \right]}_{\text{real poles}} \underbrace{\left[ \prod_{i=1}^{N_c} (z + p_i + jc_i)(z + p_i - jc_i) \right]}_{\text{complex conjugate poles}}$$

where  $N = N_r + 2N_c$  is called the *system order*.



- For a BIBO stable system, all poles [i.e. roots of characteristic polynomial  $A(z)$ ] must lie inside the unit circle.
- Systems with (dominant) poles residing inside left-hand semi-circular region exhibit forced oscillations at half the sampling frequency.
  - Very undesirable feature!

The stability of the system may be determined from the locations of the closed-loop poles in the  $z$ -plane or the roots of the closed-loop characteristic equation defined by Eq.

$$P(z) = 1 + G(z) = 0$$

as follows:

1. For the system to be stable, the closed-loop poles or the roots of the characteristic equation must lie within the unit circle in the  $z$ -domain. Any closed-loop pole outside the unit circle makes the system unstable.
2. If a simple pole lies at  $z=1$  or  $z=-1$ , then the system becomes marginally stable. Also, the system becomes marginally stable if a single pair of complex-conjugate poles lies on the unit circle in the  $z$ -domain. Any multiple closed-loop pole on the unit circle makes the system unstable.
3. Closed-loop zeros do not affect the absolute stability and therefore may be located anywhere in the  $z$ -plane. Thus, a linear time-invariant single-input–single-output discrete-time closed-loop system becomes unstable if any closed-loop poles lies outside the unit circle or any multiple closed-loop pole lies on the unit circle in the  $z$ -domain.

## Example

Determine the stability of the following system:

$$\frac{Y(z)}{R(z)} = \frac{1.658 \times 10^{-7} z^{-1} + 6.6 \times 10^{-7} z^{-2} + 1.642 \times 10^{-7} z^{-3}}{1 - 2.98z^{-1} + 2.96z^{-2} - 0.9802z^{-3}}$$

Solution:

Rearrange the transfer function in powers of  $z$  (not  $z^{-1}$ ):

$$\frac{Y(z)}{R(z)} = \frac{1.658 \times 10^{-7} z^2 + 6.6 \times 10^{-7} z + 1.642 \times 10^{-7}}{z^3 - 2.98z^2 + 2.96z - 0.9802}$$

## Solution – Roots (Cont'd)

The characteristic polynomial  $A(z)$  becomes

$$A(z) = z^3 - 2.98z^2 + 2.96z - 0.9802$$

Matlab function `roots` comes handy at this point:

$$\text{roots}([1 \ -2.98 \ 2.96 \ -0.9802])$$

Hence, the roots of  $A(z)$  are found to be

$$p_1 = 1.0525;$$

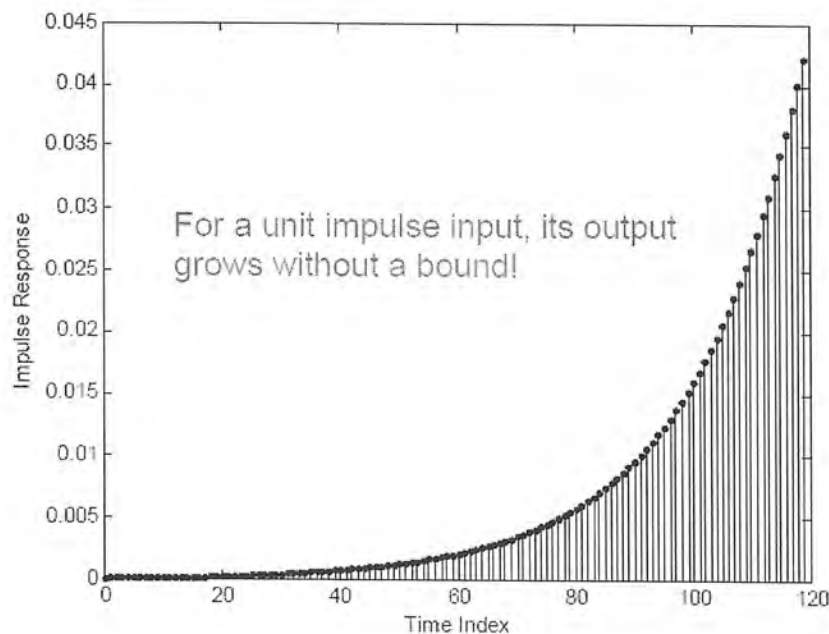
$$p_{2,3} = 0.9637 \pm j0.0499$$

Since  $p_1$  is outside the unit circle, the system is UNSTABLE!

## Solution – Matlab Script (Cont'd)

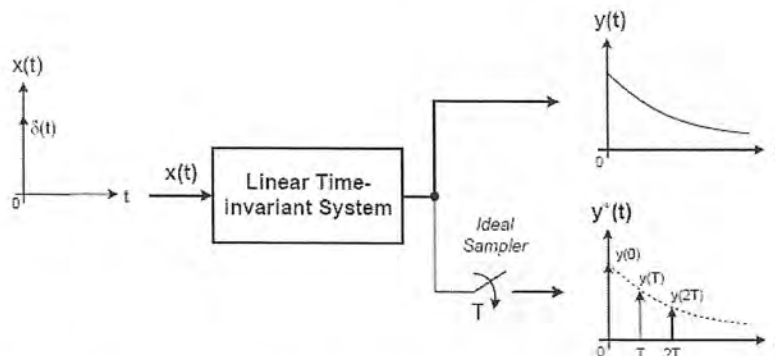
```
% *** Define the discrete-time TF
% The last argument (-1) tells Matlab that
% the sampling time is unspecified.
%
Gd = tf([1.658e-7 6.6e-7 1.642e-7],[1 -2.98 2.96 -.9802],-1);
%
% *** Calculate impulse response for 120 sampling steps...
%
[y,k] = impulse(Gd,120);
%
% *** Plot the results
%
stem(k,y,'.');
xlabel('Time Index'); ylabel('Impulse Response');
```

## Solution – Response (Cont'd)





## Testing Stability



- To test the stability, a unit impulse is applied to the system and its time response is observed.
- Let us assume that the impulse response is also sampled for sake of argument.

## Time Response of a First Order CTS

Let the transfer function of a first order system be

$$G(s) = \frac{1}{s + a}$$

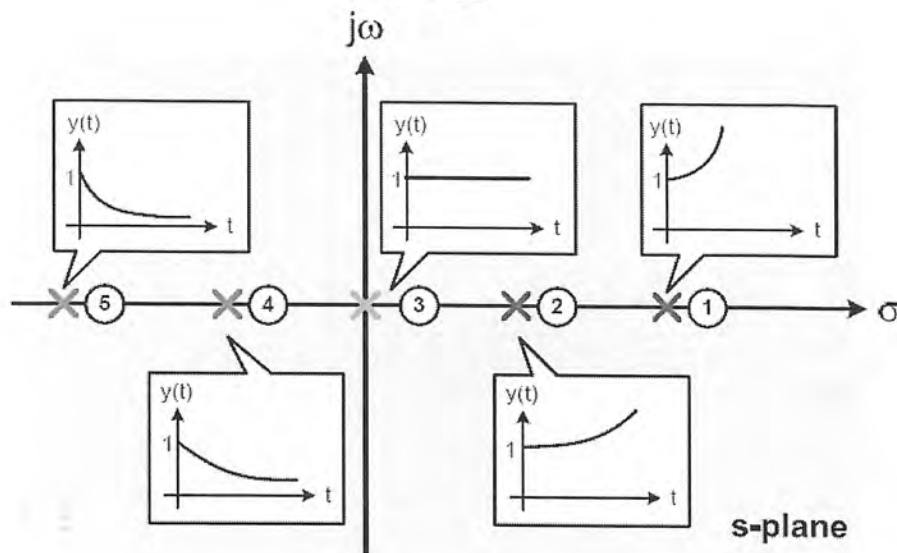
The system has one real root:  $p_1 = -a$ . Since  $X(s) = L\{\delta(t)\} = 1$ ,

$$Y(s) = G(s)X(s) = \frac{1}{s + a} \cdot 1$$

Therefore, the time response becomes

$$y(t) = L^{-1}\{Y(s)\} = L^{-1}\{G(s)\} = e^{-at}$$

## Time Response vs. Root Locus of CTS



## Time Response vs. Root Locus of DTS

Since  $y(t) = e^{-at}$ , the sampled response  $y^*(t)$  becomes

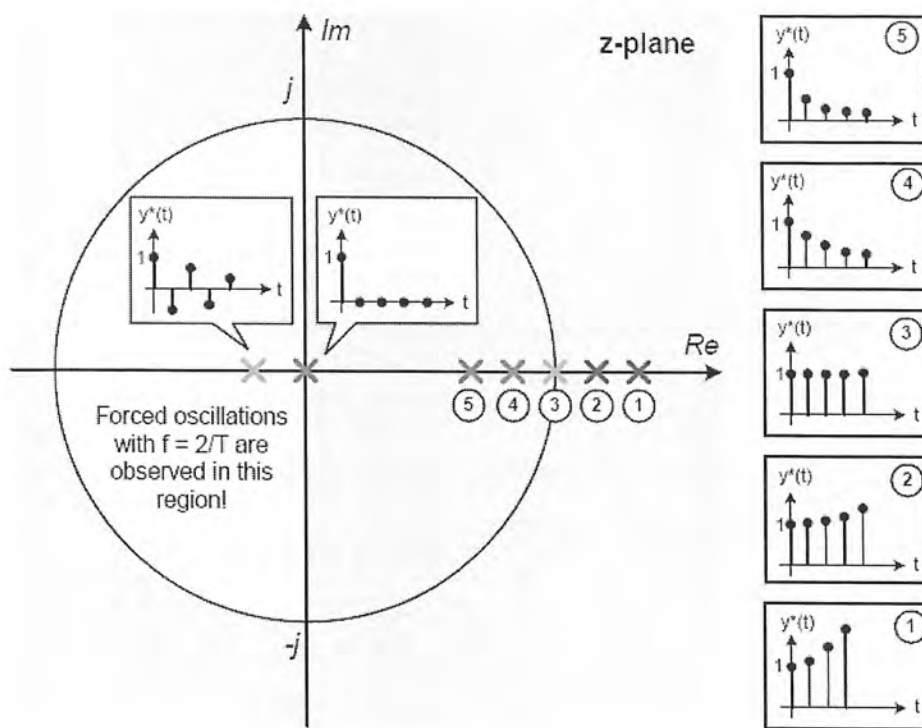
$$y^*(t) = y(kT) = e^{-akT}$$

Hence, its Z-transform leads to

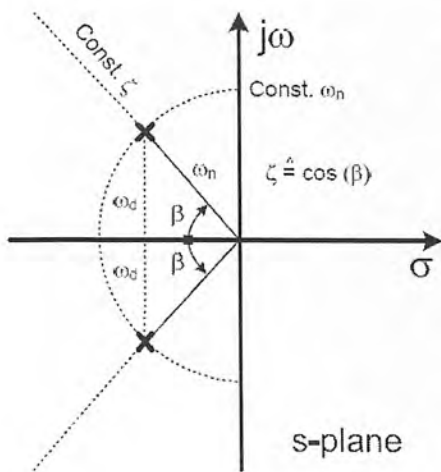
$$Y(z) = Z\{y(kT)\} = \frac{1}{1 - e^{-aT}z^{-1}} = \frac{z}{z - e^{-aT}}$$

Note that  $X(z) = 1$ . The corresponding discrete-time system has only one pole at  $p_1 = e^{-aT}$ .

## Root Locus of DTS (Cont'd)



## s-plane Analysis of Second Order CTS



The transfer function of a second-order system is expressed as

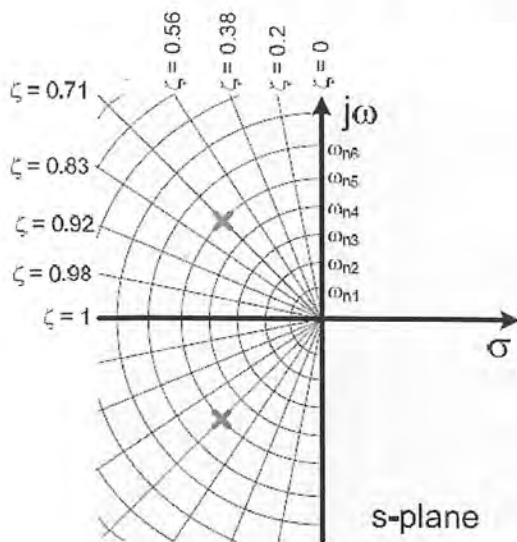
$$\frac{Y(s)}{X(s)} = \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$= \frac{K\omega_n^2}{(s + \zeta\omega_n + j\omega_d)(s + \zeta\omega_n - j\omega_d)}$$

where  $\zeta$  is the damping ratio;  $\omega_n$  is the natural frequency;  $\omega_d$  is called damped (natural) frequency:

$$\omega_d \hat{=} \omega_n \sqrt{1 - \zeta^2}$$

## s-plane Analysis (Cont'd)



- If the complex conjugate poles are located along the lines, the damping ratio does not change.
- If the poles are located on a particular semi-circle, the natural frequency is kept constant.

## z-plane Analysis of a Second Order DTS

The transfer function of a second-order discrete-time system becomes

$$\frac{Y(z)}{X(z)} = \frac{b_1 z^{-1} + b_2 z^{-2}}{1 - 2e^{-\zeta\omega_n T} \cos(\omega_d T) z^{-1} + e^{-2\zeta\omega_n T} z^{-2}}$$

$$= \frac{z^2 - 2e^{-\zeta\omega_n T} \cos(\omega_d T) z + e^{-2\zeta\omega_n T}}{z^2 - 2e^{-\zeta\omega_n T} \cos(\omega_d T) z + e^{-2\zeta\omega_n T}}$$

The roots of the characteristic polynomial are

$$p_{1,2} = e^{-\zeta\omega_n T} [\cos(\omega_d T) \pm j \cdot \sin(\omega_d T)]$$

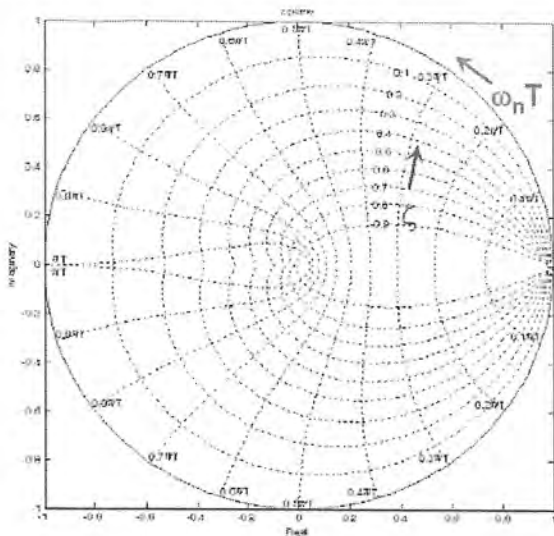
$\uparrow$   
 $e^{-\sigma T}$

$$\sigma = \zeta \omega_n$$

### Remark

To transform s-plane pole locations to the z-domain, the transformation  $z = e^{Ts}$  where  $T$  is the sampling time, is employed.

## z-plane Analysis (Cont'd)



- Mapping is slightly more complicated!
- Numbers along the radial direction shows decreasing damping ratio.
- Numbers along the perimeter of the unit circle indicate constant natural frequency.

## Matlab Script to Produce “zgrid”

```

wdT = linspace(0,pi,100)'; zeta = linspace(0,.9,10)';
p1 = zeros(100,10); p2 = p1; j = sqrt(-1);
%
% Plot constant damping contours
%
for i = 1:10
    sT1 = wdT*(-zeta(i)/sqrt(1-zeta(i)^2) + j);
    sT2 = wdT*(-zeta(i)/sqrt(1-zeta(i)^2) - j);
    p1(:,i) = exp(sT1); p2(:,i) = exp(sT2);
end
close all; plot([p1 p2]); hold on
xlabel('Real'); ylabel('Imaginary'); axis('square')
%
% Plot constant (wn*T) contours
%
zeta = linspace(0,1,100)'; wnT = linspace(pi/10,pi,10)';
for i = 1:10
    sT1 = wnT(i)*(-zeta + j*sqrt(1-zeta.^2));
    sT2 = wnT(i)*(-zeta - j*sqrt(1-zeta.^2));
    p1(:,i) = exp(sT1); p2(:,i) = exp(sT2);
end
plot([p1 p2]); zgrid % Comparison!
    
```