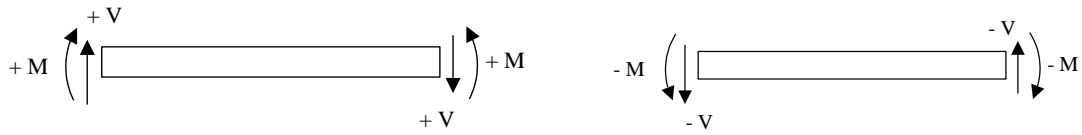
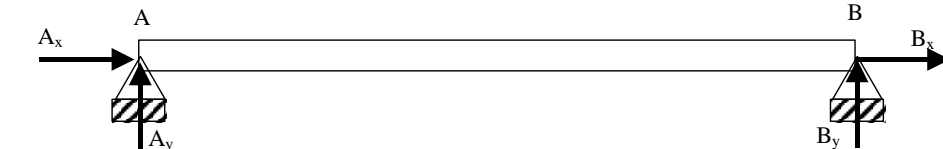


# Some Guidelines for Constructing Shear Force and Bending Moment Diagrams

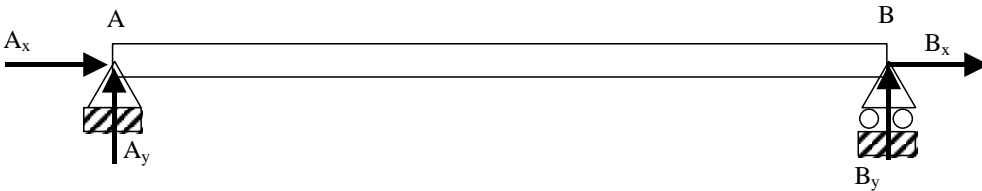
## SIGN CONVENTIONS



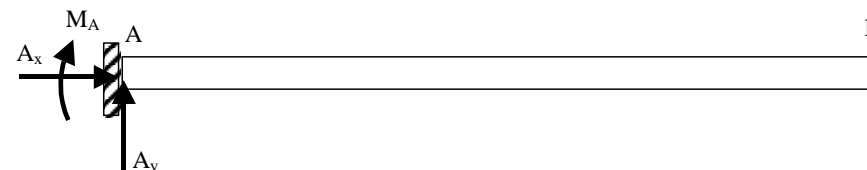
## BOUNDARY CONDITIONS



<p><b>Pinned LEFT End:</b>                  Reaction force, <math>A_x</math> is unknown.                  Reaction force, <math>A_y</math> is unknown.  <math>V</math> (shear force) = <math>A_y</math> (reaction force)  <math>M = 0</math> (unless there is an applied moment at this point)                  Deflection, <math>y = 0</math>                  Slope, <math>\theta</math> is unknown</p>	<p><b>Pinned Right End:</b>                  Reaction force, <math>B_x</math> is unknown.                  Reaction force, <math>B_y</math> is unknown.  <math>V</math> (shear force) = <math>-B_y</math> (reaction force)  <math>M = 0</math> (unless there is an applied moment at this point)                  Deflection, <math>y = 0</math>                  Slope, <math>\theta</math> is unknown</p>
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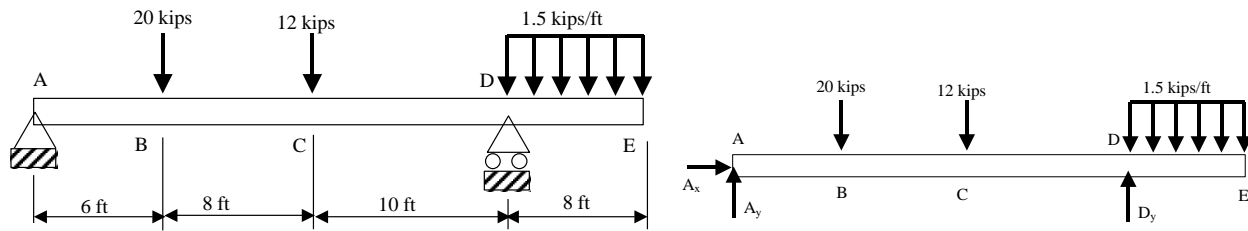
<p><b>Pinned LEFT End:</b>                  Reaction force, <math>A_x</math> is unknown.                  Reaction force, <math>A_y</math> is unknown.  <math>V</math> (shear force) = <math>A_y</math> (reaction force)  <math>M = 0</math> (unless there is an applied moment at this point)                  Deflection, <math>y = 0</math>                  Slope, <math>\theta</math> is unknown</p>	<p><b>Roller Support at Right End:</b>                  Reaction force, <math>B_x = 0</math>.                  Reaction force, <math>B_y</math> is unknown.  <math>V</math> (shear force) = <math>-B_y</math> (reaction force)  <math>M = 0</math> (unless there is an applied moment at this point)                  Deflection, <math>y = 0</math>                  Slope, <math>\theta</math> is unknown</p>
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<p><b>Clamped End:</b>                  Reaction force, <math>A_x</math> is unknown.                  Reaction force, <math>A_y</math> is unknown.                  Reaction moment, <math>M_A</math> is unknown  <math>V</math> (internal shear force) = <math>A_y</math> (reaction force)  <math>M</math> (internal bending moment) = <math>M_A</math>                  Deflection, <math>y = 0</math>                  Slope, <math>\theta = 0</math></p>	<p><b>Free End:</b>  <math>V</math> (shear force) = 0 (unless there is an applied point force at this end)  <math>M = 0</math> (unless there is an applied moment at this point)                  Deflection, <math>y</math> is unknown                  Slope, <math>\theta</math> is unknown</p>
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Example:

Draw the shear force and bending moment diagram for the following beam:



Step 1: Find the reaction forces at A and D.

Draw F.B.D.:

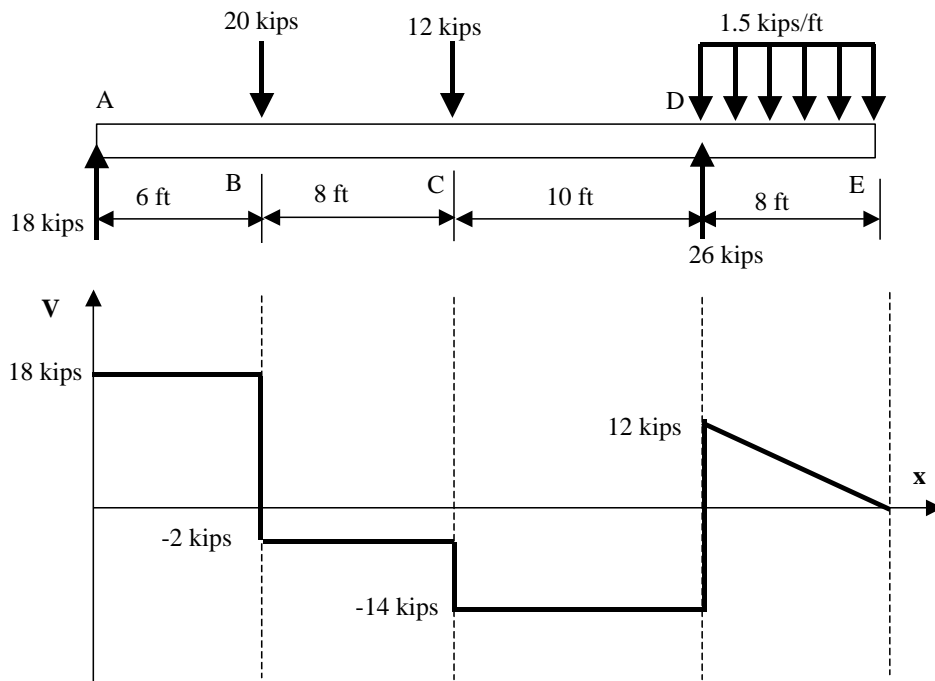
$$\sum F_x = 0, \quad A_x = 0$$

$$\sum M_A = 0, \quad (20 \text{ kips})(6 \text{ ft}) + (12 \text{ kips})(14 \text{ ft}) + (1.5 \text{ kips/ft})(8 \text{ ft})(28 \text{ ft}) - (D_y)(24 \text{ ft}) = 0$$

$$D_y = 26 \text{ kips}$$

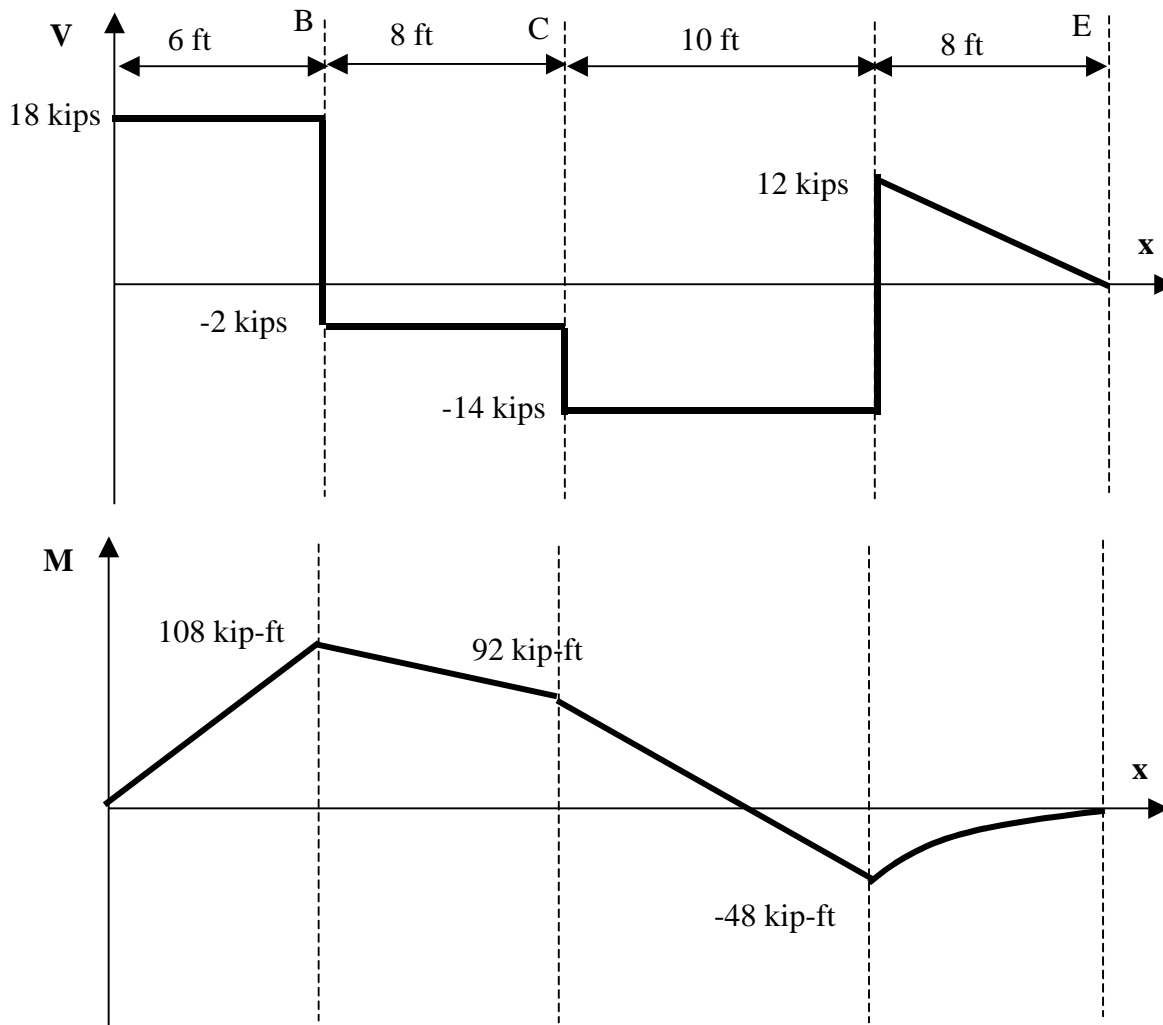
$$\sum F_y = 0, \quad A_y + D_y - 20 \text{ kips} - 12 \text{ kips} - (1.5 \text{ kips/ft})(8 \text{ ft}) = 0, \quad A_y = 18 \text{ kips}$$

### Construction of the shear force diagram



- $V = A_y = 18 \text{ kips}$  at point A.
- The shear force is constant until there is another load applied at B.
- The shear force decreases by 20 kips to  $-2 \text{ kips}$  at B because of the applied 20 kip force in the negative y direction.
- The shear force is constant until there is another load applied at C.
- The shear force decreases by 12 kips to  $-14 \text{ kips}$  at C because of the applied 12 kip force in the negative y direction.
- The shear force is constant until there is another load applied at D.
- The shear force increases by 26 kips to 12 kips at D because of the 26 kip reaction force in the positive y direction.
- The shear force decreases linearly from D to E because there is a constant applied load in the negative y-direction.
- The change in shear force from D to E is equal to the area under the load curve between D and E,  $-12 \text{ kips}$ ,  $[A_{DE} = (-1.5 \text{ kips/ft})(8 \text{ ft}) = -12 \text{ kips}]$
- The shear force at E = 0 as expected by inspection of the boundary conditions.

## Construction of the Bending Moment diagram



- a)  $M = 0$  at point A because it is a pinned end with no applied bending moment.
- b)  $M_B = M_a + (\text{the area under the shear force diagram between A and B.})$
- c)  $M_B = 0 + (18 \text{ kips})(6 \text{ ft}) = 108 \text{ kip-ft}$
- d)  $M_C = M_B + (\text{the area under the shear force diagram between B and C.})$
- e)  $M_C = 108 \text{ kip-ft} - (2 \text{ kips})(8 \text{ ft}) = 92 \text{ kip-ft}$
- f)  $M_D = M_C + (\text{the area under the shear force diagram between C and D.})$
- g)  $M_D = 92 \text{ kip-ft} - (14 \text{ kips})(10 \text{ ft}) = -48 \text{ kip-ft}$
- h)  $M_E = M_D + (\text{the area under the shear force diagram between D and E.})$
- i)  $M_E = -48 \text{ kip-ft} + \frac{1}{2} (12 \text{ kips})(8 \text{ ft}) = 0 \text{ kip-ft}$  (as expected)