



**NOTE: ANSWER FOUR QUESTIONS**

**Q1:** the O.L.T.F of a closed loop system is  $G(s)H(s) = \frac{e^{-as}(s+b)}{s^2 + 0.5s - 1}$ , (25 Marks)

**A)** Draw the state space diagram for the O.L.T.F.

**B)** the Nyquist plot for this system is shown in Fig.(1). See this figure then determine:

1. the stability of the C.L.T.F system.
2. the value of the time delay  $a$  and the constant  $b$ , if you know that  $\omega =$  for  $M=1$  and P.M =

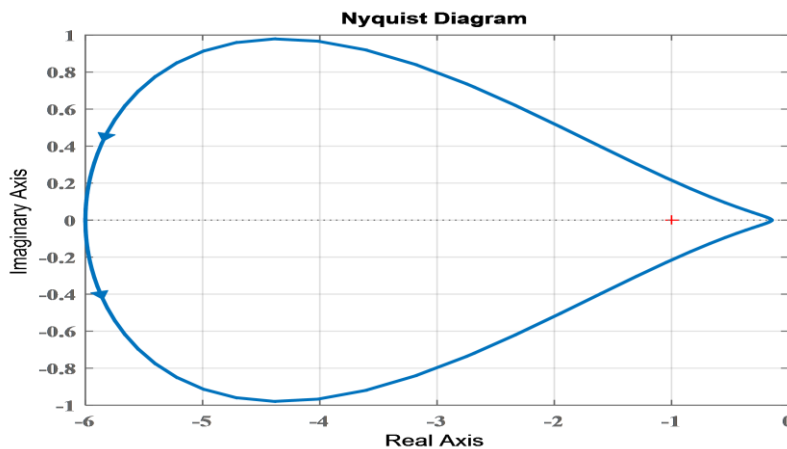


Fig.(1): Nyquist plot.

**Q2:** consider a second order unity feedback with plant  $G(s)$  and natural frequency=5 rad/sec., if unity impulse input is used to test this closed loop system for two case (critical stable response and critical damped response), then determine:

- i)** the  $G(s)$  equation for the both cases.
- ii)** the closed loop  $y(t)$  equation for the both cases.
- iii)** depending on point **ii)** determine the values of  $(y(t=t_s), e_{s,s}, \text{time constant } T, \text{ and } \sigma)$  for critical damped response case.

**Q3: A)** consider the unity f/b system with O.L.T.F: (15 Marks)

$$G(s) = \frac{K(s^2 - 2s + 4)}{(s^2 + 2s + 4)}$$

sketch the complete root locus for this system and determine the range of  $K$  for stability.

**B)** If a series PID controller is given by  $G_c(s) = 3(s+4)$ , then answer **two** from: (10 Marks)

- i)** If the controller parameters are determined by (Z-N) 1<sup>st</sup> tuning method then determine the approximate  $G_r(s)$  equation.

- ii) If the controller parameters are determined by (Z-N) 2<sup>nd</sup> tuning method then determine  $k_{cr}$  and the frequency  $\omega$ .
- iii) the controller parameters are determined by (P-Z) cancelation tuning method then determine the plant  $G(s)$  equation.

**Q4:** Consider a unity feedback system with open loop D.E given by: **(25 Marks)**

$$\ddot{y}(t) + 2\dot{y}(t) + 5y(t) = k(6\dot{u}(t) + 6u(t))$$

Then select the correct answer for **five** from the following points:

1. the O.L.T.F is: **a-** (stable 2<sup>nd</sup> order type 2)    **b-** (unstable 2<sup>nd</sup> order type 2)    **c-** (stable 4<sup>th</sup> order type 2 stable)    **d-** (unstable 4<sup>th</sup> order type 2)
2. if the input is unite sine and ( $k=1, t=2$  sec.,  $\omega=1$  rad/sec.) then the value of  $y_{s.s}(t)$  is:  
**a-** (-0.6625)    **b-** (1.8974)    **c-** (2.334)    **d-** (-4.1127)
3. according to Routh stability approach the  $k_{cr}$  and the frequency  $\omega$  at sustained oscillation are:  
**a-** ( $k_{cr}=2.34, \omega=2.15$ )    **b-** ( $k_{cr}=3.4, \omega=4.2$ )    **c-** ( $k_{cr}=6.01, \omega=2.14$ )    **d-** ( $k_{cr}=1.33, \omega=1.41$ )
4. the value of  $k$  for  $E_{s.s}=0.2$  and input  $r(t) = (4t + 2t^2)/2$  is:    **a-** (3.861)    **b-**(8.334)  
**c-** (4.645)    **d-** (5.861)
5. the suitable controller for this system is: **a-** (PID)    **b-** (PD)    **c-** (PI)    **d-** (P only)
6. the suitable tuning method for controller parameters is:  
**a-** (robust method)    **b-** (D-S method)    **c-** (P-Z method)    **d-** (C-C method)

**Q5:** Consider a unity feedback system with  $G(s) = \frac{K(s^2 + 0.8s + 0.4)}{s(1 - xs)^2}$  and data given by the

table(1), try to complete this table, determine the value of  $\omega_{s.p}, x, \omega_x$  and sketch the bode plot on Fig.(2) then determine: **(25 Marks)**

- i)** the stability of the closed loop system.
- ii)** the system gain  $k$  for gain cross-over frequency to be 0.4 rad/sec.
- iii)** the system gain  $k$  for maximum positive phase margin.

Table(1): bode data.

$\omega$	$\omega_{s.p}$	0.635	$\omega_x$	100	1000
Mag.	20	11.2	21.9	7.84	-12
$\Phi$	-437	-335	-185	-102	-91.2

Fig.(2):Bode plot.

