

Fluid in motion

The behavior of fluid in motion is very complex due to the large number of variable involved. Since viscosity introduces a great complexity to fluid flow behavior, we assume the fluid is an ideal one, this mean no viscosity and no shear stress as it flow. We also assume incompressible flow which mean that the density of fluid does not change throughout the system.

Properties of flow:

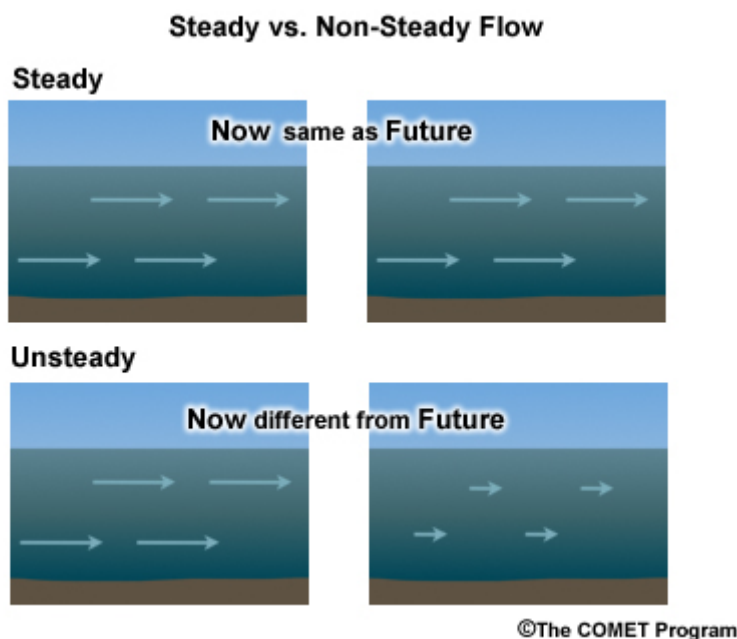
Steady flow: when there is no change in flow properties(Q, v, h) with time.

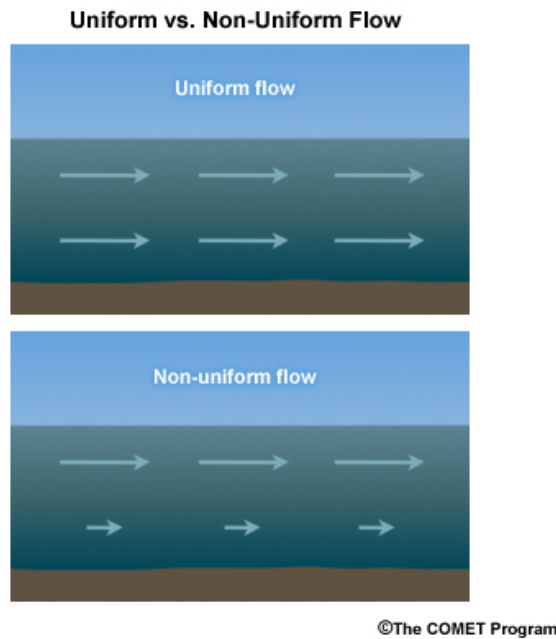
Unsteady flow: when there is change in flow properties with time.

Uniform flow: when there is no change in fluid properties with distance.

Non-uniform flow(2D): flow with two directions (x, y) and velocity distribution become parabola section with maximum value at the center and minimum at the wall of pipe.

Three-dimensions flow(3D): flow with 3 directions (x, y, z) as flow through orifice.





The continuity equation

This theorem states that mass can neither be created nor destroyed. For steady flow, the rate at which mass enters a control volume equals the rate at which mass leaves this volume. In fig.1, fluid is flowing from left to right. The pipe has two different sizes (A_1 and A_2). The volume between 1 and 2 is the control volume (CV). The rate at which the mass enters equals the rate at which the mass leaves CV.

Mass flow rate

From fig.1 let M represent the rate at which mass enters or leaves the CV, we have $M_1 = M_2$, thus for steady flow the mass flow rate at the inlet to the CV equals the mass flow rate at the exit from the CV.

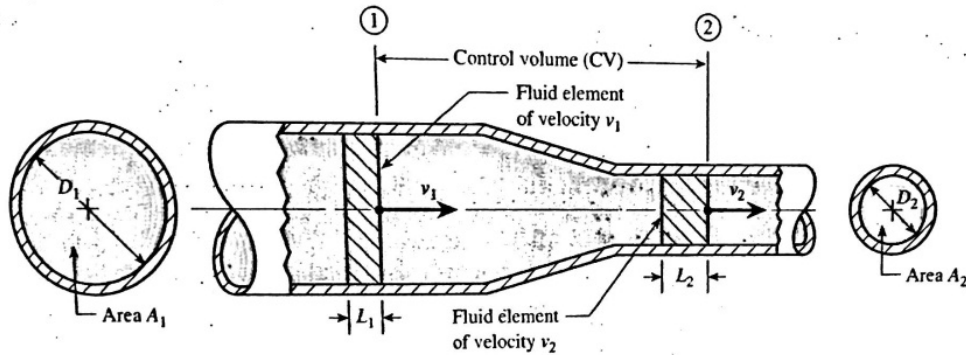


Fig.1

$$M_1 = \frac{m_1}{t} = \frac{\rho_1 A_1 L_1}{t} = \rho_1 A_1 v_1 \quad \text{and}$$

$$M_2 = \frac{m_2}{t} = \frac{\rho_2 A_2 L_2}{t} = \rho_2 A_2 v_2$$

$$\text{So} \quad \rho_1 A_1 v_1 = \rho_2 A_2 v_2$$

Weight flow rate

Since specific weight equals times the acceleration of gravity ($\gamma = \rho * g$)

The continuity equation shows that we also have equality of weight flow rates entering and leaving the CV, $W_1 = W_2$

$$\gamma_1 A_1 v_1 = \gamma_2 A_2 v_2$$

volume flow rate

for steady incompressible flow the volume flow rate is also constant represented by Q (flow), because flow = volume per time.

$$Q_1 = Q_2$$

$$\frac{\text{Vol.1}}{t} = \frac{\text{vol.2}}{t}$$

$$\frac{\text{vol}_1}{\text{vol}_2} = \frac{A_2}{A_1} = \left(\frac{D_2}{D_1}\right)^2$$

D is the diameter for circular section.

Or $Q = v \cdot A$ (v is velocity)

Examples:

Ex1:

For the fig.1 the following data are given: $D_1=4$ in. $D_2=2$ in.
 $v_1=4$ ft/s

Find the volume flow rate, fluid velocity at sec. 2, weight flow rate and mass flow rate?

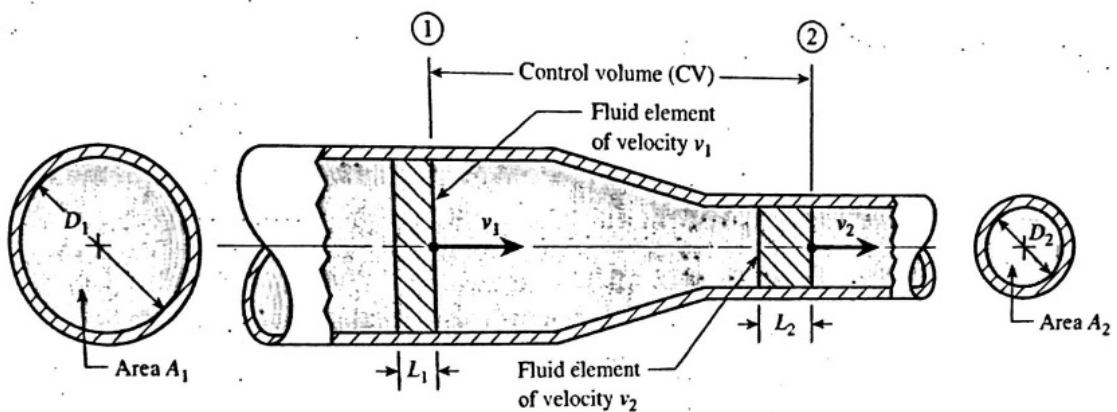


Fig.1

$$Q = Q_1 = A_1 v_1$$

$$A_1 = \frac{\pi}{4} D^2 = \frac{\pi}{4} (4/12)^2 = 0.0873 \text{ ft}^2$$

$$Q = (0.0873) (4) = 0.349 \text{ ft}^3/\text{s}$$

$$\text{b- } v_2 = v_1 (D_1/D_2)^2$$

$$= 4(4/2)^2 = 16 \text{ ft/s}$$

$$\text{c- } W = W_1 = \gamma A_1 v_1 = \gamma Q_1 = 62.4 * 0.349 = 21.8 \text{ lb/s}$$

$$\text{d- } M = M_1 = \rho A_1 v_1 = \rho Q_1 = 1.94 * 0.349 = 0.677 \text{ slug/s}$$

Ex2: A pipe line of 300 mm diameter carrying water at an average velocity 4.5 m/s branches into two pipes of 150 mm, 200mm dia., if the average velocity in the 150mm is 5/8 of the

velocity in the main pipe, find the average velocity of flow in 200 mm, and the total flow rate in the system by l/s?

$$Q=AV=Q_1+Q_2$$

$$AV=A_1V_1+A_2V_2$$

$$\frac{1}{4}\pi(0.3)^2*4.5=\frac{1}{4}\pi(0.15)^2*\frac{5}{8}*4.5+\frac{1}{4}\pi(0.2)^2*V_2$$

$$V_2=8.54 \text{ m/s}$$

$$\text{Total flow } Q=\frac{1}{4}\pi(0.3)^2*4.5$$

$$=0.318 \text{ m}^3/\text{s}$$

$$=318 \text{ l/s}$$

Ex3:

The velocity of a liquid (s.g)=1.4, in 150mm pipeline is 0.8 m/s. Calculate the rate of flow in l/s , m³/s, kg/s and KN/s?

$$Q=VA=0.8\left(\frac{\pi}{4}\right)\left(\frac{150}{1000}\right)^2=0.01414 \text{ m}^3/\text{s}=14.141 \text{ l/s}$$

$$M=\rho*Q=(1.4*1000)*0.01414=19.79 \text{ kg/s}$$

$$W=\gamma*Q=(1.4*9.8)*0.01414=0.194 \text{ KN/s}$$

Example 4: (fig. 2) Two stream of water enter the mixing chamber if the inlet velocity 80 kg/sec. and 100 kg/sec. find the outlet mass flow? And outlet weight flow?

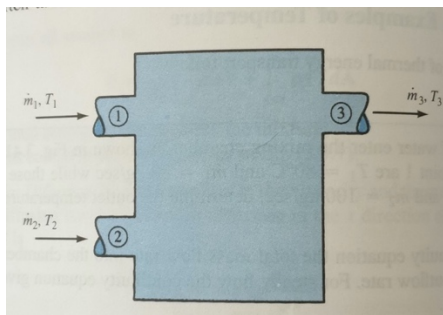


Fig. 2

Total mass flow rate = $m_1 + m_2$

$80 + 100 = 180 \text{ kg/sec.}$

$W = \text{mass} * g$

$180 * g = 180 * 9.8 = 1764 \text{ N/sec}$

Bernoulli's equation

Bernoulli's equation is based on the conservation of energy law, which states that energy can be neither created nor destroyed. The total energy possessed by a given mass of fluid can be considered to consist of three types: potential, kinetic and flow energy.

Fig.3 shows fluid flowing from left to right, the total energy by a given weight w of fluid entering CV at sta.1 and the same weight of fluid leaving CV at sta.2.

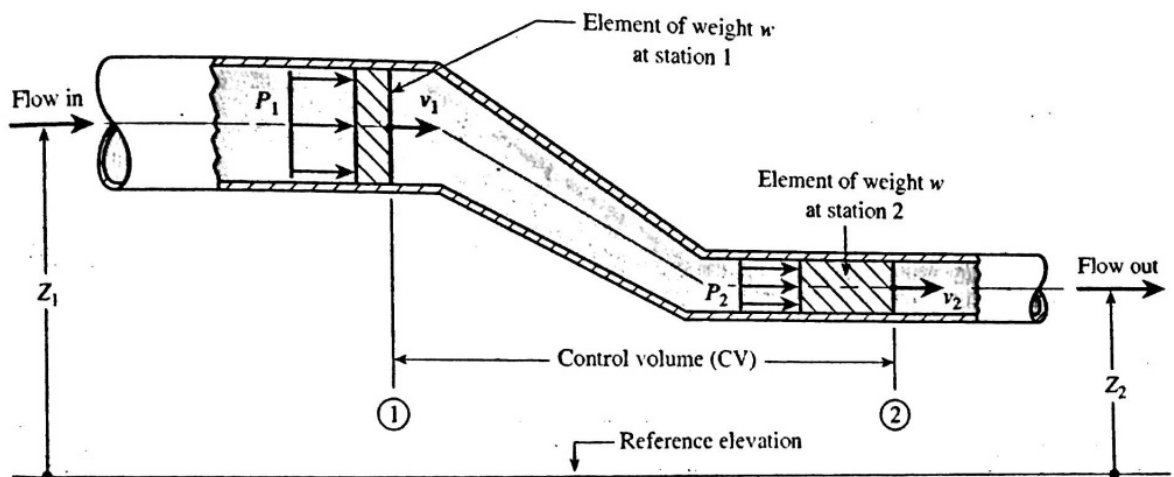


Fig.3

Potential energy: الطاقة الكامنة

The fluid element of weight w has a potential energy due to its elevation Z related to a reference plane.

$$PE = wz$$

Kinetic energy: الطاقة الحركية

The fluid element of weight w moving with a velocity

$$KE = \frac{1}{2} mv^2$$

Flow energy; طاقة الجريان

It is the amount of work that pressure accomplishes by pushing the element of weight w at sta.1 into the CV or pushing the element of weight w at sta.2 out of CV.

$$FE = \frac{Pw}{\gamma}$$

Statement of Bernoulli's equation:

Daniel Bernoulli an eighteenth century Swiss scientist, formulated his equation by noting that the total energy possessed by the fluid in CV does not change with respect to time.

Total energy in element at 1 = total energy in element at 2

$$(PE+KE+FE)_1 = (PE+KE+FE)_2$$

$$wZ_1 + \frac{1}{2} \frac{w}{g} v_1^2 + \frac{P_1 w}{\gamma} = wZ_2 + \frac{1}{2} \frac{w}{g} v_2^2 + \frac{P_2 w}{\gamma} \quad \text{divide by } w$$

$$Z_1 + \frac{v_1^2}{2g} + \frac{P_1}{\gamma} = Z_2 + \frac{v_2^2}{2g} + \frac{P_2}{\gamma}$$

$$\frac{P_1}{\gamma} + \frac{v_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{v_2^2}{2g} + Z_2$$

Z : elevation head.

$\frac{P}{\gamma}$: pressure head.

$\frac{v^2}{2g}$: velocity head

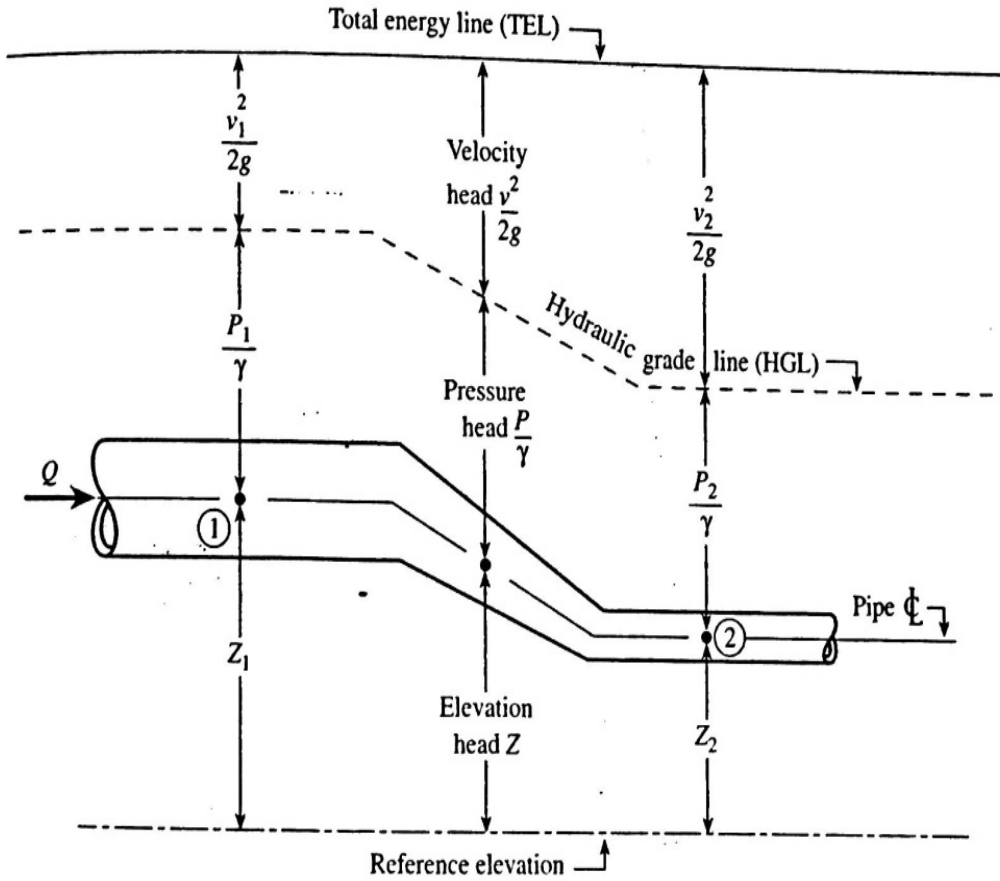


Fig.4

Ex5: For the pipe of fig.5 find P_2 if the following data are given: $p_1 = 20$ psi, $D_1 = 2$ in and $D_2 = 1.5$ in, $Q = 200$ gpm of water?

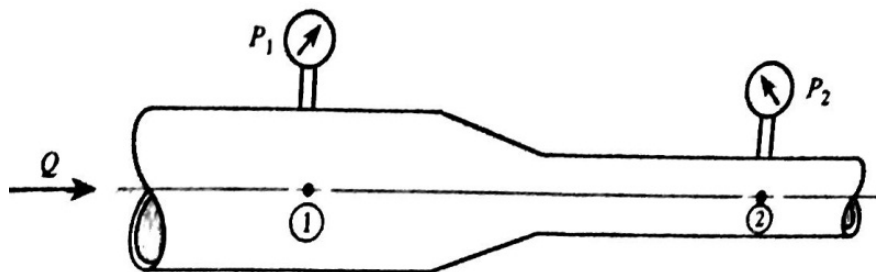


Fig.5

$$Z_1 + \frac{v_1^2}{2g} + \frac{P_1}{\gamma} = Z_2 + \frac{v_2^2}{2g} + \frac{P_2}{\gamma}$$

$$V_1 = \frac{Q}{A_1} = \frac{200 \cdot 1 \cdot 231 \cdot 1}{\frac{\pi}{4} \left(\frac{2}{12}\right)^2 \cdot 60 \cdot 1 \cdot 1728}$$

$$= 20.4 \text{ ft/s}$$

$$v_2 = v_1 (D_1/D_2)^2$$

$$\frac{20 \cdot 144}{62.4} + \frac{20.4}{2(32.2)} = \frac{P_2}{62.4} + \frac{36.2^2}{2(32.2)} + 46.2 + 6.46 = \frac{P_2}{62.4} + 20.3$$

Solving for P at 2 yields:

$$P_2 = 2020 \text{ lb/ft}^2 \text{ gage} = 14 \text{ psig.}$$

Torricelli's theorem:

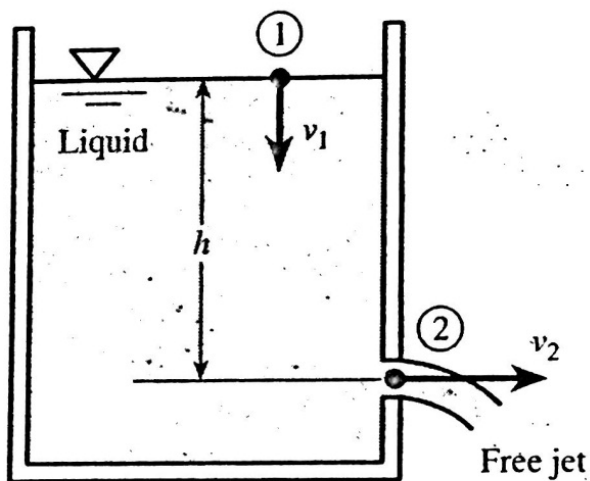


Fig.6

The velocity of free jet of fluid is equal to:

$$V = \sqrt{2gh}$$

Because:

1- $P_1 = P_2$ atmosphere pressure

2- v_1 negligible cross sec. A_1 very large compared with A_2 so

$$v_1 = \frac{A_2}{A_1} v_2 \quad (\text{very small number})$$

3- Z_2 can be taken as a zero number.

EX6: for the system in fig.6, 198 h=36 ft and diameter of side opening is 2 in find the jet velocity and the volume flow rate in gpm?

$$V_2 = \sqrt{2gh}$$

$$= \sqrt{2 * 32.2 * 36} = 48.3 \text{ ft/s}$$

$$Q = A_2 v_2 = \frac{\pi}{4} (2)^2 * 48.3 * 12 * \frac{1}{231 \text{ in}^3} * 60$$

$$= 473 \text{ gpm}$$

Ex.7

In fig. 7 whoe do the magnitude of the velocity of the three jets compare with each other?

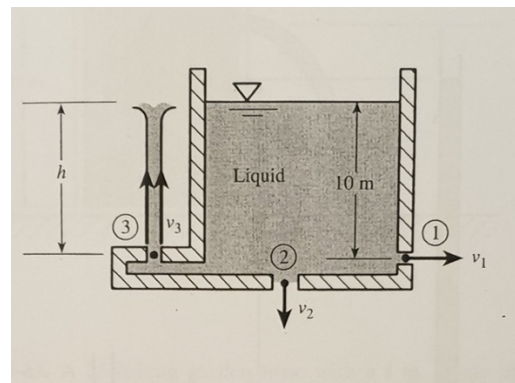


Fig. 7

From the torricillies theorem

$$V = \sqrt{2gh}$$

$$V_1 = v_2 = v_3 = \sqrt{2 * 9.8 * 10} = 14 \text{ m/s}$$

Ex. 8

In fig. 8, a free jet of water going out of tank, find the distance x the jet make contact with ground?

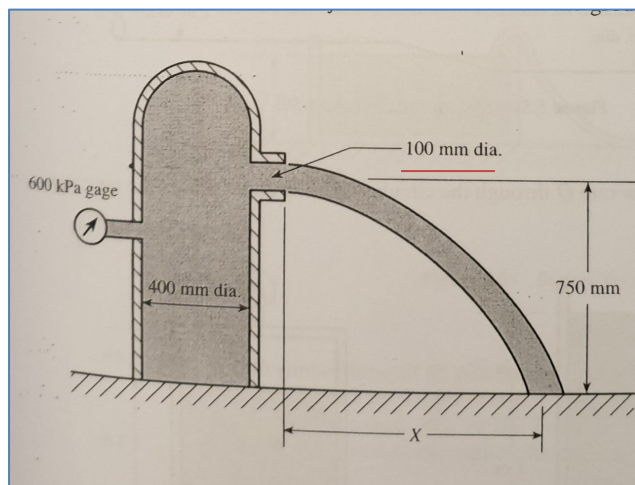


Fig. 8

$$\frac{P_1}{\gamma} + \frac{v_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{v_2^2}{2g} + Z_2$$

$$V_1 A_1 = V_2 A_2$$

$$V_1 = V_2 A_2 / A_1$$

$$\frac{P_1}{\gamma} + \frac{(V_2 A_2 / A_1)^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{v_2^2}{2g} + Z_2$$

$$\frac{600 * 1000}{9800} + \frac{(V_2 * 16)^2}{2 * 9.8} + 0 = 0 + \frac{V_2^2}{2 * 9.8} + 0$$

$$61.22 + 13.06 V_2^2 = 19.6 * V_2^2$$

$$V_2 = 3.06 \text{ m/s}$$

$$X = Vt$$

$$t = X/V$$

$$y = \frac{g}{2} t^2$$

$$t = \sqrt{(2h/g)}$$

$$t = t$$

$$\text{so } \frac{X}{V} = \frac{\sqrt{2 h / g}}$$

$$X = V * \frac{\sqrt{2 h / g}}$$

$$= 3.06 \frac{\sqrt{2 * 0.75 / 9.8}}{1} = 1.197 \text{ m}$$

The siphon:

It is a device that is used to cause a liquid to flow from one container in an upward direction downward in to a second. as shown in fig.6,

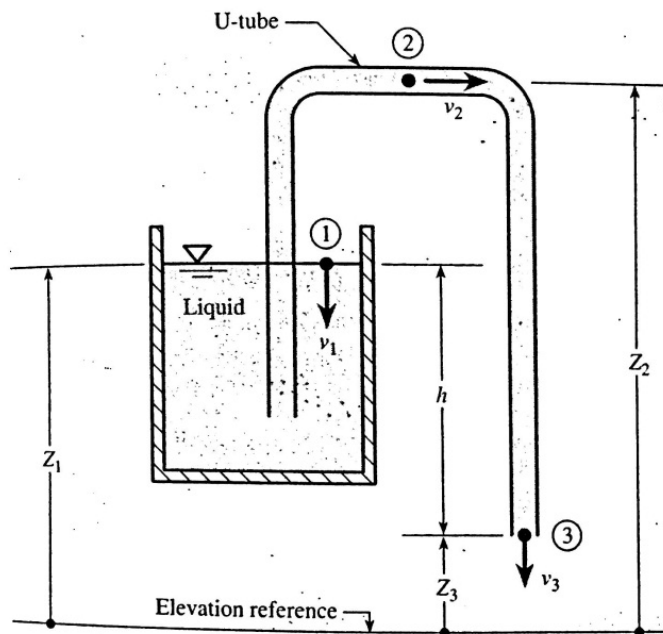


Fig.9

Point 1 lies in the free surface in the container.

Point 2 lies in the U-tube at its highest elevation.

Point 3 lies in the U-tube at the lowest elevation

The output at 3 is a free jet.

If we apply Bernoulli's eq. for points 1 & 3

$$P_1 = P_3, \quad v_1 \approx 0, \quad Z_1 - Z_3 = h, \text{ so}$$

$$V_3 = \sqrt{2gh}$$

$$\text{So } Q_3 = A_3 v_3$$

$$P_2 = \gamma(Z_1 - Z_2) + \gamma(-v_2^2/2g)$$

EX9:

Water is siphoned from a large storage tank through 50 mm diameter hose (fig.10). Find the maximum height H of a building over which the water can be siphoned?

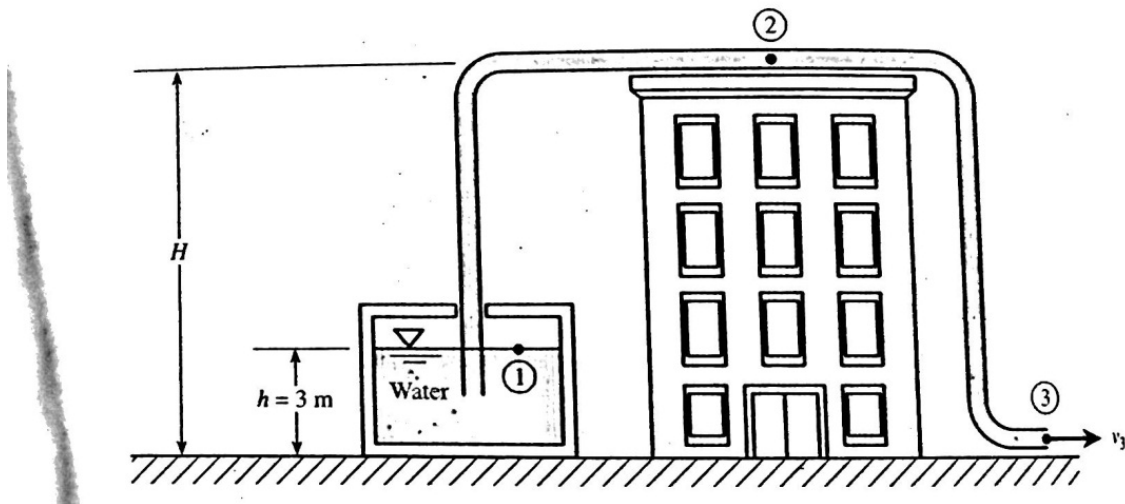


Fig.10

$$V_3 = \sqrt{2gh}$$

$$V_3 = \sqrt{2 * 9.8 * 3} = 7.67 \text{ m/s}$$

$$V_2 = v_3$$

$$P_2 = \gamma(Z_1 - Z_2) - \gamma(v_2^2/2g)$$

$$2.34 - 101 \text{ kPa} = 9800 \text{ N/m}^3(3 - H) - 9800 * \frac{7.67^2}{2 * 9.81}$$

$$-98700 = 29400 - 9800 H - 29380$$

$$H=10.1 \text{ m}$$

Ex.10

The siphon in fig.11 is filled with water and discharged at 150 L/s, find the velocity in pipe, and the pressure at point 2?

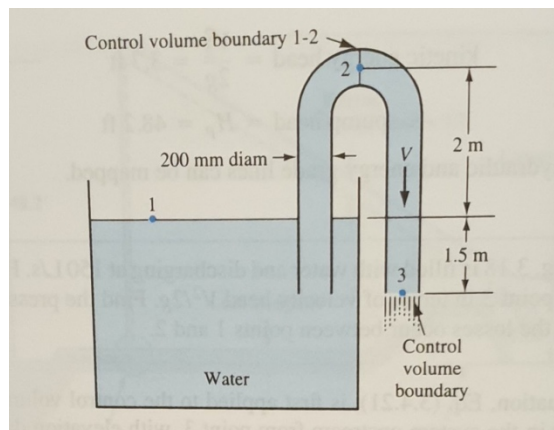


Fig.11

$$Q=V*A$$

$$V = .15 / 0.2^2 * \frac{\pi}{4} = 4.777 \text{ m/s}$$

$$\frac{P_1}{\gamma} + \frac{v_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{v_2^2}{2g} + Z_2$$

$$0+0+0 = \frac{P_2}{\gamma} + \frac{4.77^2}{2*9.8} + 2$$

$$P = 31010.03 \text{ Pa.}$$

Ex11: Water is flowing upward through the pipeline (fig.12), a manometer measures the pressure difference $p_1 - p_1$. Find the volume of flow rate?

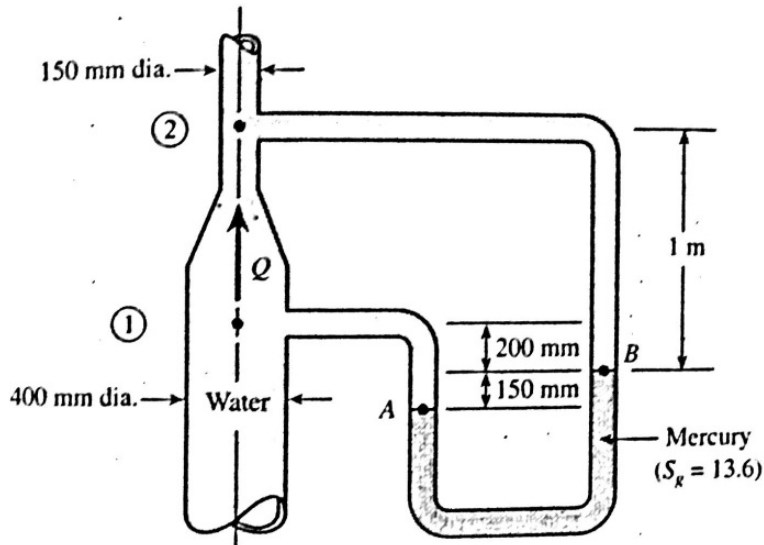


Fig.12

$$P_B = P_2 + 9800 \cdot 1 = P_2 + 9800$$

$$P_a = P_b + 9800 \cdot 13.6 \cdot 0.15$$

$$P_a = P_b + 20000 \quad P = P_2 + 29800$$

$$P_1 = P_a - 3430 = P_2 + 26370$$

$$P_1 - P_2 = 26370 \text{ pa}$$

$$Z_1 + \frac{v_1^2}{2g} + \frac{P_1}{\gamma} = Z_2 + \frac{v_2^2}{2g} + \frac{P_2}{\gamma}$$

$$A_1 v_1 = A_2 v_2$$

$$V_2 = A_1 v_1 / A_2$$

$$= \left(\frac{400}{150}\right)^2 \cdot v_1 = 7.11 v_1$$

$$\frac{v_2^2 - v_1^2}{2g} = (Z_1 - Z_2) + (P_1 - P_2) / \gamma$$

$$\frac{(7.11 v_1)^2 - v_1^2}{2 \cdot 9.81} = -(1 - 0.2) + \frac{26370}{9800}$$

$$\frac{49.6 v_1^2}{19.6} = -0.8 + 2.69 = 1.89 \text{ m}$$

$$V_1 = 0.89 \text{ m/s}$$

$$Q_1 = A_1 v_1 = \frac{\pi}{4} (0.4)^2 \cdot 0.86 = 0.108 \text{ m}^3/\text{s}$$

The pitot-static tube:

Fig.9, shows a pitot tube installed in a pipeline along with piezometer, both are pressure measuring device. Unlike, piezometer, pitot continues inward toward the centerline of the pipe. As a result the fluid that enters the inside of the pitot tube comes to a stop (stagnates), in contrast the piezometer tube allows the fluid to travel past without any change of velocity.

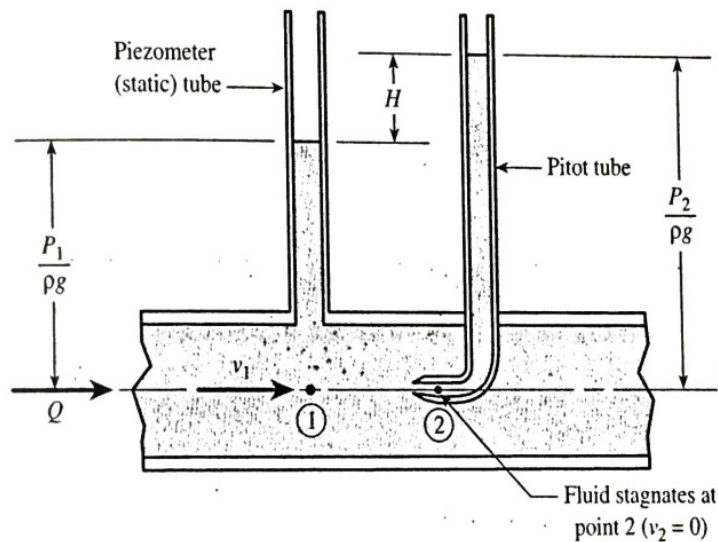


Fig.13

$$Z_1 + \frac{v_1^2}{2g} + \frac{P_1}{\gamma} = Z_2 + \frac{v_2^2}{2g} + \frac{P_2}{\gamma}$$

$$Z_1 = Z_2 \quad \text{and} \quad v_2 = 0 \quad (\text{stagnation point})$$

$$H = \frac{v_1^2}{2g}$$

Ex 12:

Determine the flow rate through the pipe?

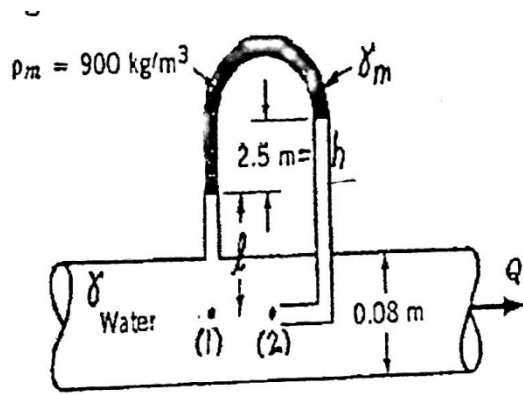


Fig. 14

$$Z_1 = Z_2 \quad v_2 = 0$$

$$\frac{v_1^2}{2g} + \frac{P_1}{\gamma} = \frac{P_2}{\gamma}$$

$$V_1 = \sqrt{\frac{2g(P_2 - P_1)}{\gamma}}$$

$$P_1 - \gamma L - \gamma_m h + \gamma(L + h) = P_2$$

$$\text{So } v_1 = \sqrt{\frac{2g \left(1 - \frac{\gamma_m}{\gamma}\right) h}{}}$$

$$= (2(9.81)(1 - 900/999)(2.5))^{0.5}$$

$$= 2.2 \text{ m/s}$$

$$Q = A_1 V_1 = \frac{\pi}{4} (0.08)^2 (2.2) = 0.0111 \text{ m}^3/\text{s}.$$

Example 13:

In the pipe in figure 15, a water manometer is connected to ends of a piezometer tube and pitot tube, find the water velocity and the flow ?

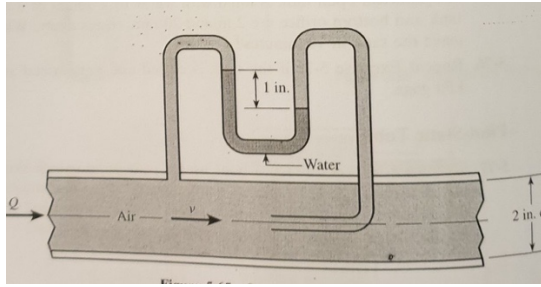


Fig.15

$$\frac{P_1}{\gamma} + \frac{v_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{v_2^2}{2g} + Z_2$$

From manometer

$$P_1 - \gamma_w h + \gamma h g^* h - \gamma_w \gamma^* h + \gamma_w h = P_2$$

$$P_1 - \gamma_w h + 13.6 * 62.4 * (1/12) - 62.4 (1/12) + \gamma_w * h = P_2$$

$$P_2 - P_1 = 65.52 \text{ lb/ft}^2$$

From Bernoulli eq.

$$\frac{65.52}{62.3} = \frac{v_1^2}{2 * 32.2}$$

$$V = 8.22 \text{ ft/s}$$

$$Q = V * A$$

$$8.22 * \frac{2^2 \pi}{12^2 4} = 1.0716 \text{ ft}^3/\text{s}$$

Venturi, nozzle and orifice flow meters:

Three additional used flow meters measuring that operate on the principles of Bernoulli eq., venturi, nozzle and orifice flow meters. They based on the principle that as the velocity of flow increases, a drop in pressure occurred. The measurement of the pressure drop can be used to indicate the flow rate.

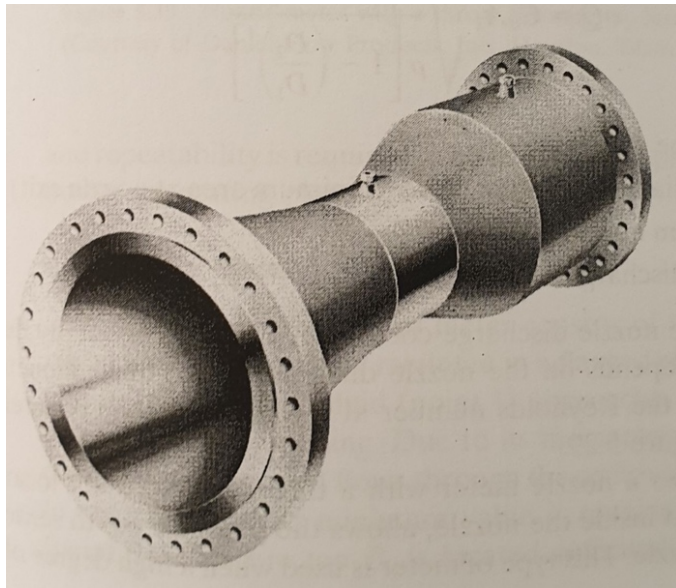


Fig. 16

Figs 16, 7, show an actual venturi meter and a venturi meter is installed to the pipeline whose flow rate is measured, venturi meter consists of a converging section followed by a constant diameter section (throat) followed by a diverging section.

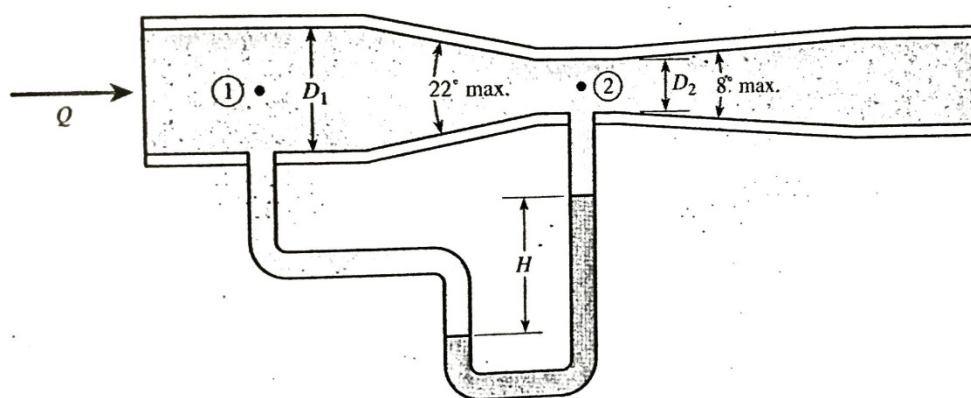


Fig. 17

$$Z_1 + \frac{v_1^2}{2g} + \frac{P_1}{\gamma} = Z_2 + \frac{v_2^2}{2g} + \frac{P_2}{\gamma}$$

$$Z_1 = Z_2$$

$$A_1 v_1 = A_2 v_2$$

$$\text{So } C_v \text{ ideal} = \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{A_2}{A_1}\right)^2\right)}}$$

To calculate the flow coefficient $C_v = \frac{V_2 \text{ actual}}{V_2 \text{ ideal}}$

$$Q = v_2 \text{ actual } A_2 = C_v \text{ ideal } A_2$$

$$Q = C_v A_2 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{A_2}{A_1}\right)^2\right)}} = C_v A_2 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{D_2}{D_1}\right)^2\right)}}$$

Fig.18, shows the nozzle meter contain a nozzle has a flange on its upstream face.

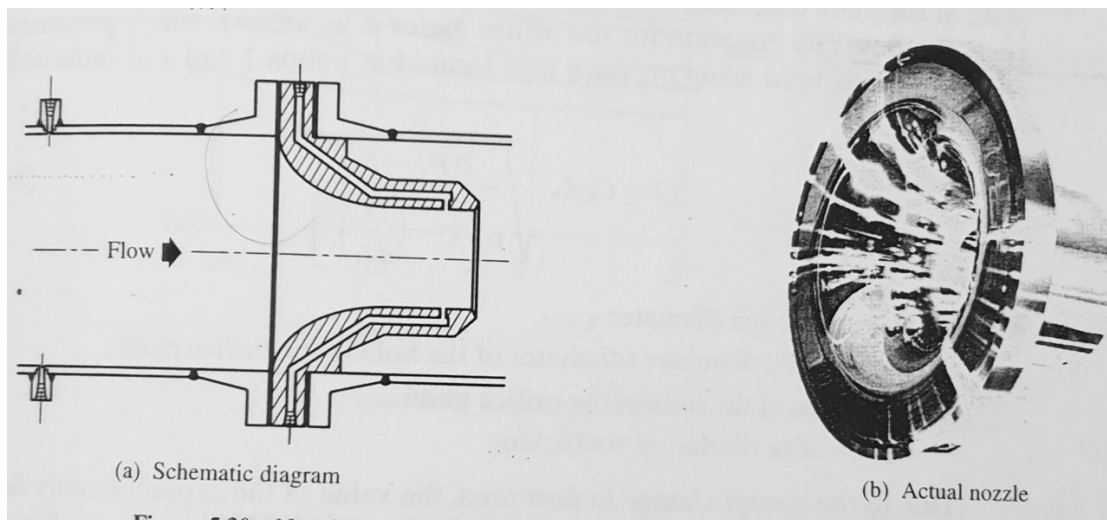


Fig.18

$$Q = C_n A_2 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{D_2}{D_1}\right)^2\right)}}$$

C_n discharge coefficient.

Fig.19, shows the orifice meter it is a simple flow meter consist of a circular plate containing a sharp edge, the upstream fluid approach the orifice it must turn inward to inter the orifice. The fluid jet cannot immediately change directions as its flow, thus the jet area continues to contract until its minimum value is obtained downstream:

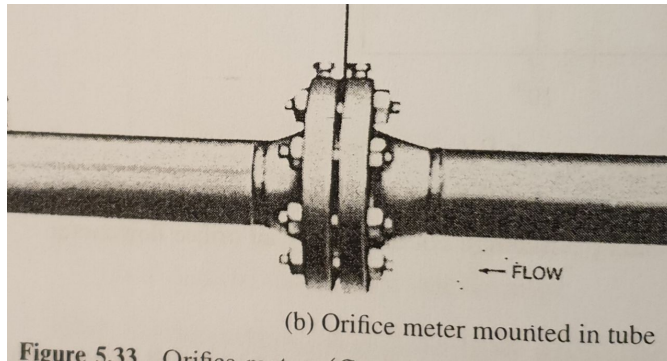


Figure 5.33 Orifice

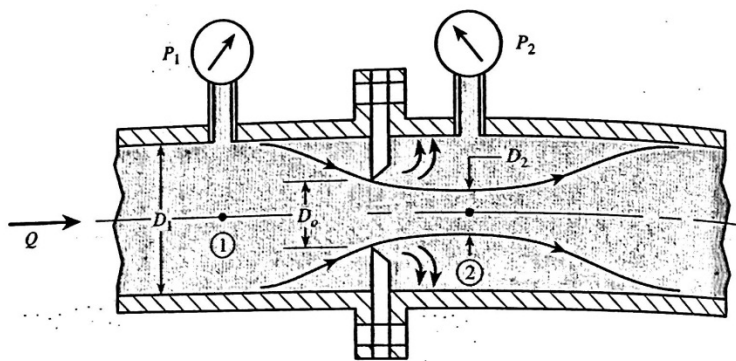


Fig.19

$$Q = C_0 A_0 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{D_0}{D_1}\right)^2\right)}}$$

A_0 : area of the hole in the orifice plate.

Ex 14:

Venture meter of fig.17, has the following data: D_1 3in, D_2 0.75 in, C_v 0.98, fluid in manometer is mercury, $P_2 - P_1 = 10$ psi, $H = 20$ in Hg, find the flow rate in gallons per minute ?

$$Q = C_v A_2 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{A_2}{A_1}\right)^2\right)}}$$

$$A_2 = \frac{\pi}{4} \left(\frac{0.75}{12}\right)^2 = 0.00307 \text{ ft}^2$$

$$\rho_{\text{water}} = 1.94 \text{ slugs/ft}^3$$

$$P_1 - P_2 = 10 * 144 = 1440 \text{ lb/ft}^2$$

$$\left(\frac{A_2}{A_1}\right)^2 = \left(\frac{0.75}{3}\right)^4 = 0.00391$$

$$Q = 0.98 * 0.00307 * \sqrt{\frac{2 * 1440}{1.94(1 - 0.00391)}} = 0.116 \text{ ft}^3/\text{s}$$

$$Q = 0.116 * 1728 * \frac{1}{231} * 60 = 52.1 \text{ gpm}$$

$$P_2 - P_1 = H (\gamma_{\text{Hg}} - \gamma_{\text{water}}) = 20 * 1/12 * (847 - 62.4) = 1308 \text{ lb/ft}^2$$

$$Q \propto \sqrt{P_1 - P_2} \quad Q = 52.1 * \sqrt{\frac{1308}{1440}} = 49.7 \text{ gpm} .$$

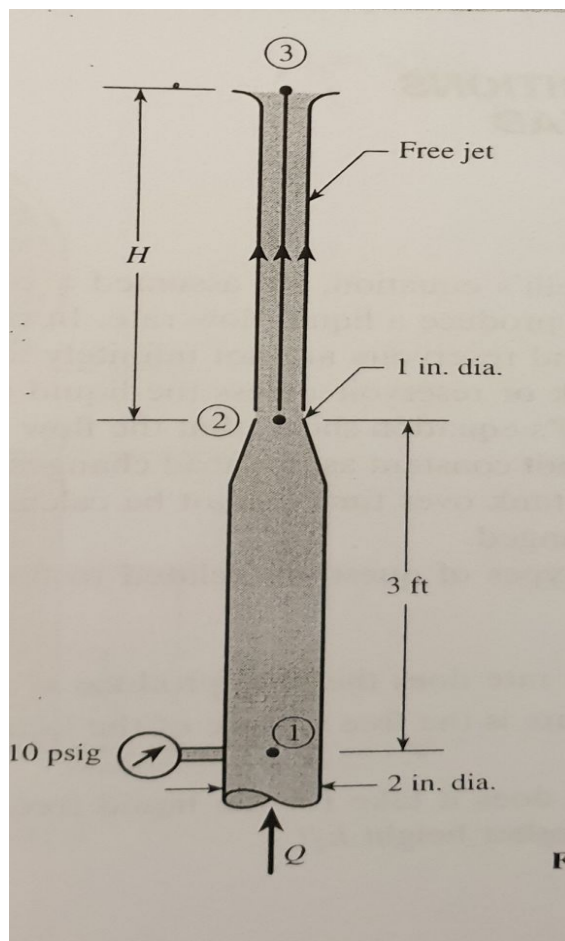
Ex. 15 An orifice flow meter consists of a 100 mm dia. pipe with 50 mm dia. sharp edge orifice. When water flow through orifice a U- tube manometer is connected indicates a differential head of 350 mm mercury, if the discharge coefficient is 0.65, find the flow?

$$Q = C_0 A_0 \sqrt{\frac{2(P_1 - P_2)}{\rho \left(1 - \left(\frac{D_0}{D_1}\right)^2\right)}}$$

$$Q = 0.62 * 0.0025 \frac{\pi}{4} \sqrt{\frac{2 * (0.53 * 13.6 * 9800)}{1000 * \left(1 - \frac{0.0025}{0.01}\right)}}$$

$$Q = 0.0135 \text{ m}^3/\text{s}$$

Extra examplas:



1- For the figure above find Q ?

2- Water flow through a venturi meter an inlet dia. = 100 mm and throat dia. = 50 mm , P at inlet = 70 kpa. And at the throat 75 mm mercury, if the discharge coefficient = 0.95 , find the flow?