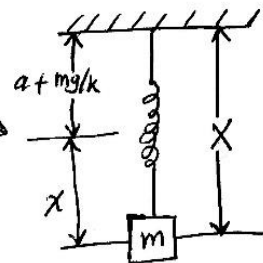
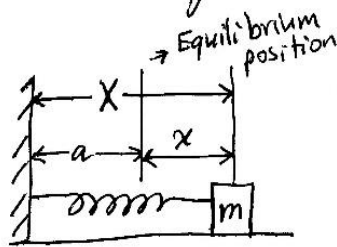


# Chapter Three (Oscillations)

## Introduction:

Everywhere around us we see systems engaged in a periodic dance: the swaying of a tree in the wind, the small oscillations of a ~~tree~~ pendulum clock, a child playing on a swing, the rise and fall of the tides. The essential feature that all these phenomena have in common is periodicity, a pattern of movement or displacement that repeats itself over and over again.

### 3-1: Linear Restoring Force. Harmonic Motion



\* a particle of mass (m) attached to a spring  
(a) horizontally  
(b) vertically.

هذه هي القوة التي تسمى القوة الخطية المسترجعة  
linear restoring force

This is a force whose magnitude is proportional to the displacement of a particle from some equilibrium position and whose direction is always opposite to that of the displacement

$$F = -k(X - a) = -kx \quad \text{Hook's Law}$$

where:

X - total length

a - unstretched (zero load) length of the spring.

x - is the displacement of the spring from its equilibrium length

k - the proportionality of constant called stiffness

The total force acting on the particle is:

$$F = -k(x-a) + mg$$

(47)  
the positive direction is  
(downward motion)

The diff. equation of motion for the horizontal case is:

$$F = -kx \quad \text{or} \quad F = m \ddot{x}$$

of linear oscillator

$$-kx = m \ddot{x}$$

the diff. eq. of the harmonic oscillator

$$\Rightarrow \boxed{m \ddot{x} + kx = 0} \quad \text{--- (1) linear diff. eq. with const. coeff.}$$

where,  $m$  &  $k$  are two const. mass & stiffness const.

To solve the above equation we shall employ the trial method in which the function  $(Ae^{qt})$  is the trial solution, where  $(q)$  is a constant to be determined.

So,

$$x = Ae^{qt} \rightarrow \dot{x} = Aqe^{qt} \rightarrow \ddot{x} = Aq^2 e^{qt} \quad \text{--- (2)}$$

sub. eqs. (2) in eq (1) we get:

$$m \frac{d^2}{dt^2} (Ae^{qt}) + kAe^{qt} = 0$$

$$m \frac{d^2}{dt^2} (Ae^{qt}) + kAe^{qt} = 0$$

$$[mAq^2 e^{qt} + kAe^{qt} = 0] \quad \div Ae^{qt}$$

$$mq^2 + k = 0 \rightarrow q^2 = \frac{-k}{m} \rightarrow q = \pm \sqrt{\frac{-k}{m}}$$

$$= \pm i \sqrt{\frac{k}{m}} = \pm i \omega_0$$

where  $i = \sqrt{-1}$  and  $\omega_0 = \sqrt{\frac{k}{m}}$

$x = Ae^{i\omega t}$

for a linear diff. eqs., solutions are additive. so the general solution of eq① is:

$x = A_+ e^{+i\omega t} + A_- e^{-i\omega t} \dots \dots I$

where:  $e^{+i\omega t} \rightarrow$  is an outgoing wave.

$e^{-i\omega t} \rightarrow$  is an incoming wave.

using Euler formula:

$e^{ix} = \cos x + i \sin x, e^{-ix} = \cos x - i \sin x$

$x = A^+ (\cos \omega t + i \sin \omega t) + A^- (\cos \omega t - i \sin \omega t)$

$= A^+ \cos \omega t + i A^+ \sin \omega t + A^- \cos \omega t - i A^- \sin \omega t$

$= \frac{2A^+}{a} \cos \omega t + \frac{2A^-}{b} \sin \omega t$

$x = a \cos \omega t + b \sin \omega t \dots \dots II \left[ x \frac{\cos \omega t}{\cos \omega t} \right] \text{ or } \left[ x \frac{\sin \omega t}{\sin \omega t} \right]$

$= a \cos \omega t + b$

$x = A \cos(\omega t + \theta_0) \dots \dots III$

$x = A \sin(\omega t + \theta_0)$

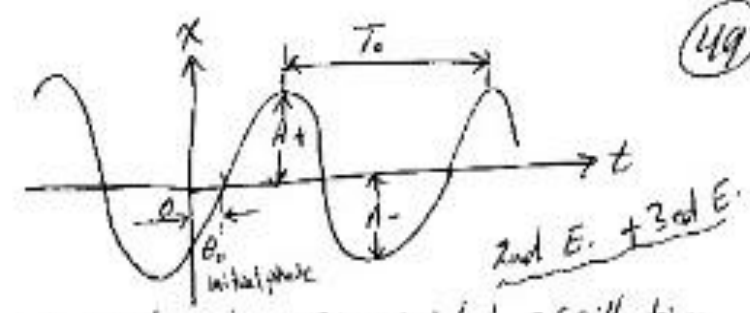
where:  $A = (a^2 + b^2)^{1/2}, \theta_0 = -\tan^{-1} \left( \frac{b}{a} \right)$  (Initial phase)

A - is amplitude  
 $\omega$  - angular freq.

where all three above equations are solutions of eq①

$$T_0 = \frac{2\pi}{\omega_0} = 2\pi \sqrt{\frac{m}{k}}$$

① period: the time for which the product  $(\omega t)$  increases by just  $(2\pi)$

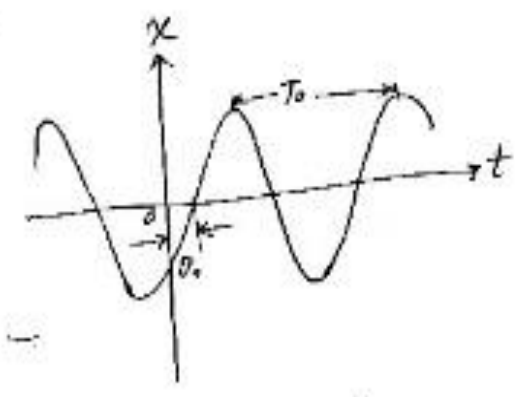


The motion is a sinusoidal oscillation of the displacement  $x$

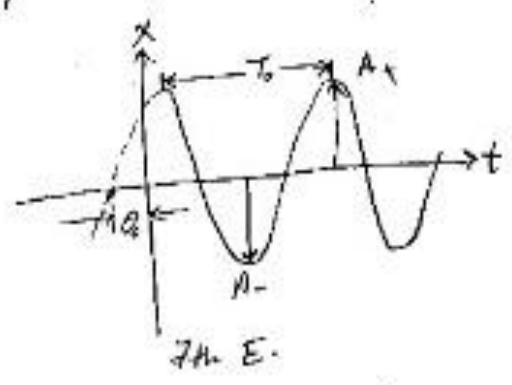
② the linear frequency of oscillation ( $f_0$ ) is defined as the number of cycles in unit time, therefore:

(radians per second)  $\rightarrow \omega_0 = 2\pi f_0$

(cycles per second)  $f_0 = \frac{1}{T_0} = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$



$x = A \sin(\omega_0 t + \theta_0)$   
equation of simple harmonic oscillator



$x = A \cos(\omega_0 t + \theta_0)$   
simple harmonic oscillator

3-2: Energy considerations in Harmonic Motion:

المعادلة الطاقة في الحركة التوافقية

Consider a particle moving under a restoring force  $F = -kx$ .  
Let us calculate the work ( $W$ ) done by an external force ( $F_a$ ) in moving the particle from the equilibrium position ( $x=0$ ) to some position  $x$ .

$$F = -kx \quad \text{--- (1)}$$

$$F_a = -F$$

$$= -(-kx) = kx$$

$$W = \int_0^x F_a(x) dx = \int_0^x (kx) dx = \frac{1}{2} k x^2 \quad \text{--- (2)}$$

قوة مساوية والتغلب  
المطبوع من قبل ( $F_a$ ) على الناظر لتعريفه خارج ازاحة الاوتار. فإذا ما انجزت  
هذه القوة الشغل المثلث وتثبت الناظر عند موضع معين (الكرن لانه  
عند الاوتار ( $F_a$ ) فان نصف الشغل سيقام محزوناً في الناظر على شكل  
طاقة كما انه يتناسب كما لا يخفى:

$$V(x) = W = \frac{1}{2} k x^2 = E_p \quad \text{--- (3)}$$

Total Spring energy:

$$E = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} k x^2 \quad \text{--- (4)}$$

We can now solve for the velocity (as a function of displacement):

$$\dot{x}^2 = \frac{2}{m} \left[ E - \frac{1}{2} k x^2 \right]$$

$$\dot{x} = \left[ \frac{2E}{m} - \frac{kx^2}{m} \right]^{1/2} \quad \text{--- (5)}$$

This can be integrated to give  $t(x)$  as a function of  $x$ :

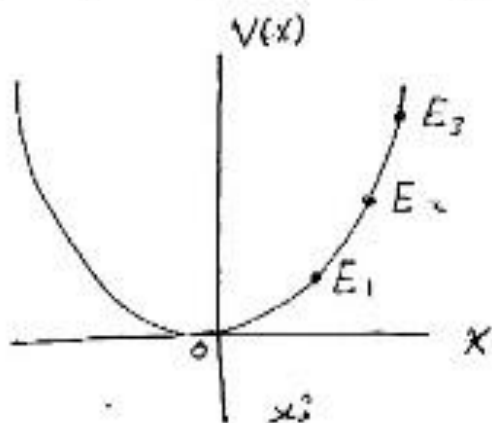
$$\dot{x} = \frac{dx}{dt} = \sqrt{\frac{2E}{m} - \frac{kx^2}{m}} \rightarrow t = \int_0^x \frac{dx}{\sqrt{\frac{2E}{m} - \frac{kx^2}{m}}}$$

(E1) لغرض حساب الشغل اللازم يجب توفير مشتقة المعاد والدالة هي:

$$\frac{k}{m} x$$

$$t = \sqrt{\frac{m}{k}} \cos^{-1}\left(\frac{x}{A}\right) + C$$

where:  $A = \sqrt{\frac{2E}{k}}$



للمذبون البسيط، تتبدل بين حالة متصلة أثناء ازدياد الزيادة  $V(x)$  عند موضع الاتزان (0) بينما أو سياراً والتي هي عبارة عن مجموع طاقة كينماتي وحركية، ما لم يصل إلى أقصى ازدياد (وهي السرعة) وهذا لا يتكون من لحاقته كما أنه متقطع. في حين أن نقطة الاتزان تكون الطاقة ولكنها صفر. وهذا هو التفسير الكلاسيكي لسقوط الطاقة في المذبذب البسيط المتوافق البسيط.

(1)  $E = \frac{1}{2} m v_{\max}^2 = \frac{1}{2} k A^2$   
 في الحالة عند نقطة الاتزان

$\therefore v_{\max}^2 = \frac{2E}{m}$

We have:  $A^2 = \frac{2E}{k} \rightarrow E = \frac{kA^2}{2}$

$\therefore v_{\max}^2 = \frac{\frac{kA^2}{2}}{m} = \frac{kA^2}{2m}$

$\Rightarrow v_{\max} = \sqrt{\frac{k}{m}} A$  Maximum velocity for the harmonic oscillator.

(2) Let:  $\frac{E}{\text{lower point}} = E_{\text{higher point}} \rightarrow \frac{1}{2} m v_{\max}^2 = \frac{1}{2} k A^2 \rightarrow A^2 = \frac{m}{k} v_{\max}^2$   
 $\therefore A = \sqrt{\frac{m}{k}} v_{\max} = \frac{v_{\max}}{\omega}$

We also see from the energy equation that the maximum value of the speed, which we call  $v_{max}$ , occurs at  $x = 0$ . Accordingly, we can write

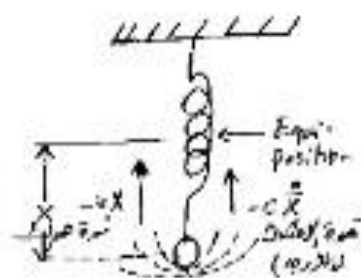
$$E = \frac{1}{2} m v_{max}^2 = \frac{1}{2} k A^2 \quad (3.3.7)$$

As the particle oscillates, the kinetic and potential energies continually change. The constant total energy is entirely in the form of kinetic energy at the center, where  $x = 0$  and  $\dot{x} = \pm v_{max}$ , and it is all potential energy at the extrema, where  $\dot{x} = 0$  and  $x = \pm A$ .

على شبيهه بهاله النابض غير المرصع السابق لو اعتبرنا ان هناك وسط لزج يحيط بالنابض فان حركة النابض لن تكون حرة ولتكون المطرقة (القوة المصدية) لن تكون هي العنصر في حساب معادله الحركة بل يجب اعتبار القوة الاخرى الموجودة وهي القوة المؤخرة (Retarding Force)

$$F = -kx \quad \text{--- (1)} \quad F = -c\dot{x} \quad \text{--- (2)}$$

where  $c$  is const. of resistance  
 is Equation of Motion then:  
 viscous retarding force (varying linearly with the speed)



The damped harmonic oscillator

$$-kx - c\dot{x} = m\ddot{x} \quad \text{--- (3)}$$

or  $m\ddot{x} + c\dot{x} + kx = 0$  --- (4) The diff eq. of Motion for the damped harmonic oscillator

وحتى نحل معادله الحركة لنستخدم التجربة (trial sol) باسما  $x = Ae^{qt}$   $\text{exp.}$  فنظرفنا ان الزاوية  $q$   $\text{--- (5)}$

$$x = Ae^{qt} \quad \text{--- (5)}$$

$$\dot{x} = qAe^{qt}$$

$$\ddot{x} = q^2 Ae^{qt}$$

بمستعمل المعادله  $x$  من معادله (3) في معادله الحركة فنحصل:

$$mq^2 Ae^{qt} - cq Ae^{qt} - kAe^{qt} = 0 \quad [ \div Ae^{qt} ]$$

$$mq^2 - cq - k = 0 \quad \text{--- (6)}$$

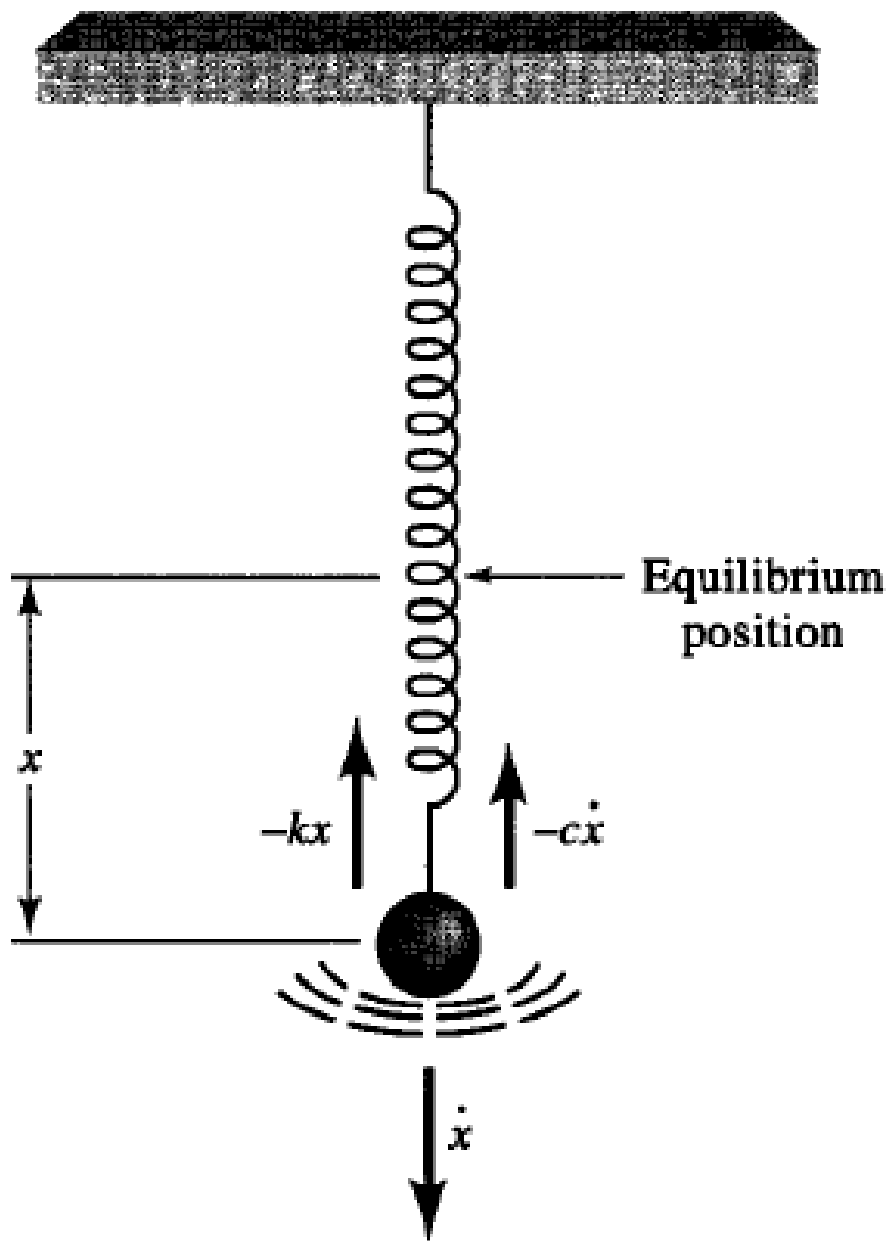
the roots are given by the well-known quadratic formula:

$$q = \frac{-c \pm (c^2 - 4mk)^{1/2}}{2m} \quad \text{--- (7)}$$

In case of  $(c^2 > 4mk)$  and  $c^2 = 4mk$ ,  $q$  will be real and negative (overdamping) (موتناقلة) (critical damping) (موتناقلة)

so the motion is non oscillatory. (غير توافقية)





من معادله المستقر، فإن (q) من الممكن ان تكتب ثلاث اشكال من القيم اعتمادا على قيمة ثابت المقاومة (C) وهي كالآتي:

q - Cases according to resistance constant (C):

Case I,

If  $c^2 > 4mk$ , then q will be real and negative and the motion will be nonoscillatory (overdamping)

$$q = - \begin{cases} \delta_1 \\ \delta_2 \end{cases} \rightarrow x = \begin{cases} A_1 e^{-\delta_1 t} \\ A_2 e^{-\delta_2 t} \end{cases}$$

so the general solution for displacement is:

$$x = A_1 e^{-\delta_1 t} + A_2 e^{-\delta_2 t} \quad \dots 8$$

Case II,

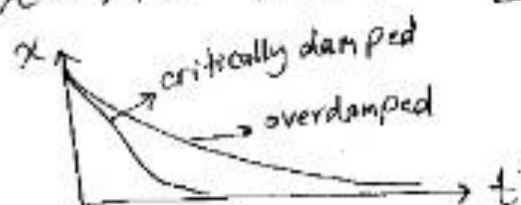
If  $c^2 = 4mk$ , the q will also be real and negative and the motion also nonoscillatory but (critical damped)

$$q = \frac{-c}{2m} \quad \dots 9$$

∴  $\delta = \delta_1 = \delta_2 = \gamma$  and we have:  $-\delta = q$

∴  $\gamma = - \left( \frac{-c}{2m} \right) = \frac{c}{2m}$ , sub. in eq. 8, we get:

$$x = A_1 e^{-\gamma t} + A_2 e^{-\gamma t} = e^{-\gamma t} (A_1 + A_2) \quad \dots 9$$



coefficients. Let  $D$  be the differential operator  $d/dt$ . We “operate” on  $x$  with a quadratic function of  $D$  chosen in such a way that we generate Equation 3.4.4:

$$\left[ D^2 + 2\gamma D + \omega_0^2 \right] x = 0 \quad (3.4.5a)$$

We interpret this equation as an “operation” by the term in brackets on  $x$ . The operation by  $D^2$  means first operate on  $x$  with  $D$  and then operate on the result of that operation with  $D$  again. This procedure yields  $\ddot{x}$ , the first term in Equation 3.4.4. The operator equation (Equation 3.4.5a) is, therefore, equivalent to the differential equation (Equation 3.4.4). The simplification that we get by writing the equation this way arises when we factor the operator term, using the binomial theorem, to obtain

$$\left[ D + \gamma - \sqrt{\gamma^2 - \omega_0^2} \right] \left[ D + \gamma + \sqrt{\gamma^2 - \omega_0^2} \right] x = 0 \quad (3.4.5b)$$

The operation in Equation 3.4.5b is identical to that in Equation 3.4.5a, but we have reduced the operation from second-order to a product of two first-order ones. Because the order of operation is arbitrary, the general solution is a sum of solutions obtained by setting the result of each first-order operation on  $x$  equal to zero. Thus, we obtain

$$x(t) = A_1 e^{-(\gamma - q)t} + A_2 e^{-(\gamma + q)t} \quad (3.4.6)$$

where

$$q = \sqrt{\gamma^2 - \omega_0^2} \quad (3.4.7)$$

The student can verify that this is a solution by direct substitution into Equation 3.4.4. A problem that we soon encounter, though, is that the above exponents may be real or complex, because the factor  $q$  could be imaginary. We see what this means in just a minute.

There are three possible scenarios:

- |                    |                  |
|--------------------|------------------|
| I. $q$ real $> 0$  | Overdamping      |
| II. $q$ real $= 0$ | Critical damping |
| III. $q$ imaginary | Underdamping     |

**I. Overdamped.** Both exponents in Equation 3.4.6 are real. The constants  $A_1$  and  $A_2$  are determined by the initial conditions. The motion is an exponential decay with two different decay constants,  $(\gamma - q)$  and  $(\gamma + q)$ . A mass, given some initial displacement and released from rest, returns slowly to equilibrium, prevented from oscillating by the strong damping force. This situation is depicted in Figure 3.4.2.

**II. Critical damping.** Here  $q = 0$ . The two exponents in Equation 3.4.6 are each equal to  $\gamma$ . The two constants  $A_1$  and  $A_2$  are no longer independent. Their sum forms a single constant  $A$ . The solution degenerates to a single exponential decay function. A completely general solution requires two different functions and independent constants to satisfy the boundary conditions specified by an initial position and velocity. To find a solution with two independent constants, we return to Equation 3.4.5b:

$$(D + \gamma)(D + \gamma)x = 0 \quad (3.4.8a)$$

Case III

(54)

If  $c^2 < 4mk$ , then  $q$  will be complex, the real part of its ~~soluti~~ value gives an oscillatory motion and the case is called (underdamped) <sup>is</sup> ~~defi~~

$$q = \frac{-c \pm \sqrt{c^2 - 4mk}}{2m} = \frac{-c \pm \sqrt{c^2 \frac{4m^2}{4m^2} - 4mk \frac{4m^2}{4m^2}}}{2m}$$

$$= \frac{-c \pm \sqrt{4m^2 \left( \frac{c^2}{4m} - \frac{k}{m} \right)}}{2m} = \frac{-c \pm 2m \sqrt{\frac{c^2}{4m^2} - \frac{k}{m}}}{2m}$$

we have:  $\gamma = \frac{c}{2m}$  ,  $\omega_0 = \frac{k}{m}$

$$\therefore q = \frac{-c \pm 2m \sqrt{\gamma^2 - \omega_0^2}}{2m} = \frac{-c}{2m} \pm \frac{2m \sqrt{\gamma^2 - \omega_0^2}}{2m}$$

$$= -\gamma \pm \sqrt{\gamma^2 - \omega_0^2} = -\gamma \pm \sqrt{-(\omega_0^2 - \gamma^2)}$$

$$= -\gamma \pm \sqrt{-1} \sqrt{\omega_0^2 - \gamma^2} = -\gamma \pm i \sqrt{\omega_0^2 - \gamma^2}$$

$$q_{1,2} = -\gamma \pm i\omega_1$$

where:  $\omega_1 = \sqrt{\omega_0^2 - \gamma^2}$

Also:  $q_1 = -\gamma + i\omega_1$  ,  $q_2 = -\gamma - i\omega_1$

Thus:  $q = \begin{cases} -\gamma + i\omega_1 = q_1 \\ -\gamma - i\omega_1 = q_2 \end{cases}$

The displacement then:

$$x = A_1 e^{(-\gamma + i\omega_1)t} + A_2 e^{(-\gamma - i\omega_1)t}$$

$$\Rightarrow x = e^{-\gamma t} [A_1 e^{i\omega_1 t} + A_2 e^{-i\omega_1 t}]$$

-- (10)

Upon using Euler formula:  $e^{iu} = \cos u + i \sin u$ !

$$x = e^{-\gamma t} (a \sin \omega_1 t + b \cos \omega_1 t)$$

where:  $a = i(A_+ - A_-)$

$b = -A_+ + A_-$

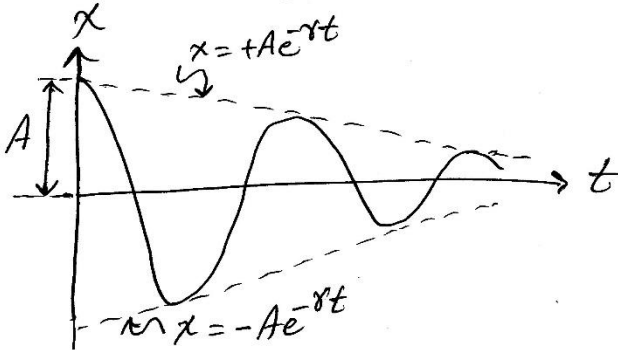
or:

$$x = A e^{-\gamma t} \cos(\omega_1 t + \phi_0)$$

where:  $A = (a^2 + b^2)^{1/2}$

$\phi_0 = -\tan^{-1}(b/a)$

The displacement  $x$  is real, its amplitude ( $Ae^{-\gamma t}$ ) decays exponentially with time.



3-4 Energy Consideration for DHO!  
 اعتبار الطاقة الكلية الوافقة للمذبذب

$$E_t = E_p + E_k$$

$$= \frac{1}{2} k x^2 + \frac{1}{2} m \dot{x}^2 \quad \dots \textcircled{C}$$

To find the time rate of change of ( $E$ ), we have to differentiate ( $E_t$ ) w.r.t. time:

$$\frac{dE_t}{dt} = \frac{1}{2} k (\dot{x} x + x \dot{x}) + \frac{1}{2} m (\dot{x} \dot{x} + x \ddot{x})$$

$$= (k\bar{x} + m\ddot{\bar{x}})\dot{\bar{x}} \quad \text{--- (2)}$$

We have the eq. of motion for the harmonic oscillator damped

$$-k\bar{x} - c\dot{\bar{x}} = m\ddot{\bar{x}}$$

$$m\ddot{\bar{x}} + k\bar{x} = -c\dot{\bar{x}}$$

or  $(k\bar{x} + m\ddot{\bar{x}}) = -c\dot{\bar{x}} \quad \text{--- (3)}$

Substituting eq (3) within eq (2), we can write:

$$\frac{dE_t}{dt} = (-c\dot{\bar{x}})(\dot{\bar{x}}) = -c(\dot{\bar{x}})^2$$

This eq. represents the rate at which energy ( $E_t$ ) dissipated (تبدد) into heat by friction (الاحتكاك) for DHO.

### 3-5 Forced Harmonic Motion. Resonance

الحركة التوافقية الجبرية. والرنين  
في هذا الموضوع نتناول DHM والتي تُقاد بواسطة قوة توافقية  
خارجية (Ext.) أي أن القوة المسلطة تتغير جيبياً (sinusoidally) مع الزمن.

$$F_{ext.} = F_0 \cos(\omega t + \theta) = F_0 e^{i(\omega t + \theta)}$$

where:  $F_0$  - amplitude  
 $\omega$  - angular frequency

القوة المؤثرة على الجسم لأن هو  $F = -kx$  --- (1)

$$F = -c\dot{x} \quad \text{--- (2)}$$

$$F = F_0 e^{i(\omega t + \theta)} \quad \text{--- (3)}$$

The diff. eq. of motion is therefore :

$$-k\bar{x} - c\dot{\bar{x}} + F_{ext} = m\ddot{\bar{x}}$$

$m\ddot{x} + c\dot{x} + kx = F_{ext} = F_0 e^{i(\omega t + \theta)}$  Eq. of Motion for the Forced Harmonic Oscillator

معامل جيبى بوجود قوة خارجية

The suggested solution eq. is:

$$x = A e^{i(\omega t + \theta')} \rightarrow \dot{x} = i\omega A e^{i(\omega t + \theta')} = i\omega \bar{x}$$

$$\ddot{x} = i^2 \omega^2 A e^{i(\omega t + \theta')} = -\omega^2 \bar{x}$$

من جوفين معادله الكلاسيكية (الاولى والثانية) في معادله الحركة

$$= m\omega^2 A e^{i(\omega t + \theta')} + ic\omega A e^{i(\omega t + \theta')} + kA e^{i(\omega t + \theta')} = F_0 e^{i(\omega t + \theta)}$$

$e^{-i(\omega t + \theta')}$

$$-m\omega^2 A + ic\omega A + kA = F_0 (e^{i\omega t} \cdot e^{i\theta} \cdot e^{-i\omega t} \cdot e^{-i\theta'})$$

في المعادله الكلاسيكية

$$= F_0 e^{i(\theta - \theta')}$$
$$= F_0 [\cos(\theta - \theta') + i \sin(\theta - \theta')]$$

where:  $\theta - \theta' = \varphi$ , which is the angle of phase diff. Separation of real and imaginary terms, we get:

$$-m\omega^2 A + kA = F_0 \cos \varphi$$

$$A(k - m\omega^2) = F_0 \cos \varphi \quad \dots *$$

and  $c\omega A = F_0 \sin \varphi \quad \dots **$

$i$  is eliminated  
في المعادله

deviding eq ~~\*\*~~ on eq ~~\*~~ we get:

$$\frac{c\omega A}{A(k - m\omega^2)} = \frac{F_0 \sin \varphi}{F_0 \cos \varphi}$$

بسته به لیب  
واکس (m)

$$\tan \phi = \frac{c\omega}{k - m\omega^2} = \frac{\frac{c}{m}\omega}{\frac{k}{m} - \frac{m\omega^2}{m}}$$

We have:  $\delta = \frac{c}{2m}$  (damped frequency)

$$\circ \circ \quad \boxed{2\delta = \frac{c}{m}} \quad \boxed{\omega_0 = \sqrt{\frac{k}{m}}}$$

$$\phi = \tan^{-1} \left( \frac{2\gamma\omega}{\omega_0^2 - \omega^2} \right)$$

$\tan \phi = \frac{2\delta\omega}{\omega_0^2 - \omega^2}$  The difference in phase between the applied driving force and the steady-state response

Solving for  $A$  (the amplitude of the steady-state oscillation), yields: (squaring both eq (\*) & (\*\*), then adding)

الاعادة  
المعادلات  
للمثلث  
المثلث  
المثلث  
المثلث

$$A^2(k - m\omega^2)^2 = F_0^2 \cos^2 \phi$$

$$c^2 \omega^2 A^2 = F_0^2 \sin^2 \phi$$

$$A^2(k - m\omega^2)^2 + c^2 \omega^2 A^2 = F_0^2 (\cos^2 \phi + \sin^2 \phi)$$

$$= F_0^2$$

$$A^2[(k - m\omega^2)^2 + c^2 \omega^2] = F_0^2$$

تقسيم الطرفين  
بـ m

$$\circ \circ \quad A = \frac{F_0}{\sqrt{(k - m\omega^2)^2 + c^2 \omega^2}} = \frac{\frac{F_0}{m}}{\sqrt{\left(\frac{k}{m} - \frac{m\omega^2}{m}\right)^2 + \frac{c^2}{m^2} \omega^2}}$$

$$\boxed{A = \frac{F_0/m}{\sqrt{(\omega_0^2 - \omega^2)^2 + 4\delta^2 \omega^2}}}$$

is the initial  
--- (#) steady state Amplitude  
oscillation

The equation that relating the amplitude ( $A$ ) to the impressed driving frequency ( $\omega$ ).



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في معادله الرنين، فان اعظم قيمه لـ (A) تتحقق عند  
عندما يقبل  $(\omega = \omega_r)$  اي عند تردد الرنين Resonance  
frequency ولا يباد فيه تردد الرنين بحيث ان نشأه  
معادله الرنين بالنسبه للتردد من اولى النسخه بالفرق.

$$\frac{dA}{d\omega} = \frac{d}{d\omega} \left[ \frac{F_0/m}{\sqrt{(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2}} \right] = 0$$

$$= \frac{F_0}{m} \frac{d}{d\omega} [(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2]^{-\frac{1}{2}} = 0$$

$$= \frac{F_0}{m} \frac{d}{d\omega} [\omega_0^4 - 2\omega_0^2 \omega^2 + \omega^4 + 4\gamma^2 \omega^2]^{-\frac{1}{2}} = 0$$

$$= \frac{F_0}{m} \left[-\frac{1}{2}\right] [\omega_0^4 - 2\omega_0^2 \omega^2 + \omega^4 + 4\gamma^2 \omega^2]^{-\frac{1}{2}-1}$$

$$[2\omega_0^2 \omega - 2(2)\omega_0^2 \omega + 4\omega^3 + (4)(2)\gamma^2 \omega] = 0$$

$$= \frac{-F_0}{2m} \frac{4\omega\omega_0^2 + 4\omega^3 + 8\gamma^2 \omega}{[(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2]^{3/2}} = 0$$

$$= \frac{-F_0}{2m} \frac{2\omega[\omega_0^2 + \omega^2 + 2\gamma^2]}{[(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2]^{3/2}} = 0 \rightarrow \boxed{2\omega = ?}$$

$$\left[ \frac{(-F_0)(2\omega)}{m} \right]$$

$$= \frac{-F_0}{m} \frac{2\omega[\omega^2 - \omega_0^2 + 2\gamma^2]}{[(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2]^{3/2}} = 0$$

$$\frac{\omega^2 - \omega_0^2 + 2\gamma^2}{[(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2]^{3/2}} = 0$$

بما ان الكسر = صفر، فان المقام لا يمكن ان يساوه صفر وذلك يعني ان  
(البسط = صفر)

$$\omega^2 - \omega_0^2 + 2\gamma^2 = 0$$

$$\rightarrow \omega^2 = \omega_0^2 - 2\gamma^2$$

for  $\boxed{\omega = \omega_r = (\omega_0^2 - 2\gamma^2)^{1/2}}$  Resonance Frequency  
---(##)  
(frequency for max.  
amplitude)

In case of weak damping, that is, ( $c \ll 2\sqrt{mk}$ ) or ( $\gamma \ll \omega_0$ )

then:  $\boxed{\omega_r \approx \omega_0}$

أي سيكون التردد (الرنين) مقارباً لـ  $\omega_0$  لـ تردد  
المذبذب الحر (بدون احتكاك).

$$\omega_r = (\omega_0^2 - 2\gamma^2)^{1/2}$$

Binomial Theorem for expansion:

$$(\omega_0^2 - 2\gamma^2)^{1/2} \approx \boxed{(1+x)^{1/2} = 1 + \frac{1}{2}x + \frac{\frac{1}{2}(\frac{1}{2}-1)}{2!}x^2 + \dots}$$

$$(\omega_0^2 - 2\gamma^2)^{1/2} =$$

\* The steady-state amplitude at the resonant frequency which we shall call ( $A_{max.}$ ) is obtained as the following:

$$A = \frac{F_0/m}{\sqrt{(\omega_0^2 - \omega^2)^2 + 4\gamma^2\omega^2}} \quad \text{---(##) (amplitude at steady-state condition)}$$

$$\text{eq(##): } \omega^2 = \omega_0^2 - 2\gamma^2 \quad \rightarrow \quad 2\gamma^2 = \omega_0^2 - \omega^2$$

----- ① ----- ②

sub. eq ① & ② in eq(##) yields:

$$A_{max.} = \frac{F_0/m}{\sqrt{(2\gamma^2)^2 + 4\gamma^2(\omega_0^2 - 2\gamma^2)}} = \frac{F_0/m}{\sqrt{4\gamma^4 + 4\gamma^2\omega_0^2 - 8\gamma^4}}$$

$$A_{\max} = \frac{F_0/m}{\sqrt{4\gamma^2\omega_0^2 - 4\gamma^4}} = \frac{F_0/m}{\sqrt{4\gamma^2(\omega_0^2 - \gamma^2)}}$$

$$\Rightarrow A_{\max} = \frac{F_0/m}{2\gamma\sqrt{\omega_0^2 - \gamma^2}} \rightarrow A \text{ (at } \omega_1)$$

For weak damping:  $\gamma \ll \omega_0$

$$A_{\max} = \frac{F_0}{2\gamma m \omega_0}$$

$$\gamma = \frac{c}{2m} \rightarrow c = 2\gamma m$$

$$\therefore \boxed{A_{\max} = \frac{F_0}{c\omega_0}} \text{ steady-state amplitude at the resonant frequency.}$$

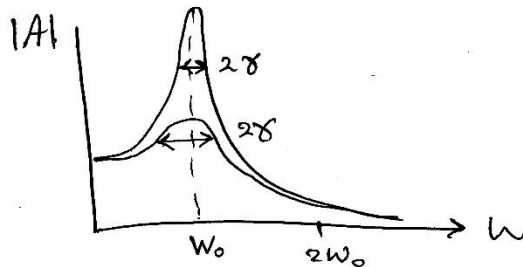
$$A = \frac{A_{\max} \gamma}{\sqrt{(\omega_0 - \omega)^2 + \gamma^2}} \text{ (relation between amplitude and max. Amplitude) at weak damping.}$$

This equation shows that, when  $|\omega_0 - \omega| = \gamma$ , then:

$$A = \frac{A_{\max} \gamma}{\sqrt{\gamma^2 + \gamma^2}} = \frac{A_{\max} \gamma}{\gamma\sqrt{2}}$$

$$\Rightarrow \boxed{A^2 = \frac{1}{2} A_{\max}^2}$$

\* من المعادلة  $|\omega_0 - \omega| = \gamma$   
 يتضح أن  $(\gamma)$  تمثل العرض  
 لمخارج الرنين. مقياس



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Another way of designating the sharpness of the resonance peak is in terms of parameter ( $Q$ ) called the quality factor of a resonant system. It is defined as:

$$Q = \frac{W_r}{2\gamma}$$

or, for weak damping:  $Q \approx \frac{W_0}{2\gamma}$

Thus the width ( $\Delta W$ ) at the half-energy points is approximately:

$$\Delta W = 2\gamma \approx \frac{W_0}{Q}$$

$$\because \omega = 2\pi f \rightarrow \Delta \omega = \Delta 2\pi f = 2\pi \Delta f$$

also:  $\omega_0 = 2\pi f_0$

$$\therefore \frac{\Delta \omega}{\omega_0} = \frac{2\pi \Delta f}{2\pi f_0} = \frac{\Delta f}{f_0} \approx \frac{1}{Q}$$

eg. (I.P. 71) (H.W.)

① A particle of mass ( $m$ ) is attached to a spring of stiffness  $k$ . The damping is such that ( $\gamma = \omega_0/4$ ). Find the ~~natural~~ resonance frequency, and the damped oscillator, and quality factor

Solution

$$W_r = (\omega_0^2 - 2\gamma^2)^{1/2} = (\omega_0^2 - \frac{2\omega_0^2}{16})^{1/2}$$

$$= \omega_0 \sqrt{\frac{7}{8}} = \sqrt{\frac{k}{m}} \sqrt{\frac{7}{m}}$$

for the resonance frequency in angular measure. The quality factor is given by:

$$Q = \frac{W_r}{2\gamma} = \frac{\omega_0 (\frac{7}{8})^{1/2}}{2(\omega_0/4)} = 2\sqrt{\frac{7}{8}} = 1.87$$

② If the applied frequency is ( $\omega_0/2$ ) for the above oscillator, find the phase angle  $\phi$ .

Solution

$$\tan \phi = \frac{2\gamma \omega}{\omega_0^2 - \omega^2} = \frac{2(\frac{\omega_0}{4})(\frac{\omega_0}{4})}{\omega_0^2 - \frac{\omega_0^2}{4}} = \frac{\frac{1}{4} \omega_0^2}{\omega_0^2 (1 - \frac{1}{4})}$$

$$\tan \varphi = \frac{1/4}{3/4} = \frac{1}{3}$$

$$\rightarrow \varphi = \tan^{-1}\left(\frac{1}{3}\right) = 18.5^\circ$$

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